

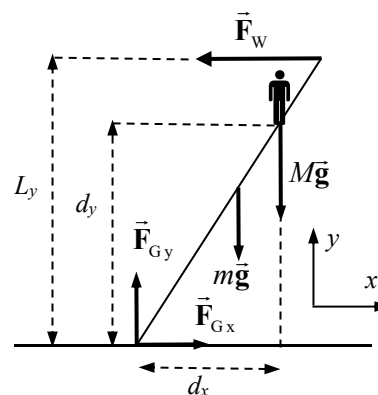
STATIC EQUILIBRIUM; ELASTICITY AND FRACTURE

Responses to Questions

1. If the object has a net force on it of zero, then its center of mass does not accelerate. But since it is not in equilibrium, it must have a net torque and therefore an angular acceleration. Some examples are:
 - A compact disk in a player as it comes up to speed, after it has just been inserted.
 - A hard drive on a computer when the computer is first turned on.
 - A window fan immediately after the power to it has been shut off.
 - The drum of a washing machine while it is speeding up or slowing down.
2. The bungee jumper is not in equilibrium, because the net force on the jumper is not zero. If the jumper were at rest and the net force were zero, then the jumper would stay at rest by Newton's first law. The jumper has a net upward force when at the bottom of the dive, and that is why the jumper is then pulled back upward.
3. The meter stick is originally supported by both fingers. As you start to slide your fingers together, more of the weight of the meter stick is supported by the finger that is closest to the center of gravity, so the torques produced by the fingers are equal and the stick is in equilibrium. The other finger feels a smaller normal force, and therefore a smaller frictional force, so the stick slides more easily and moves closer to the center of gravity. The roles switch back and forth between the fingers as they alternately move closer to the center of gravity. Your fingers will eventually meet at the center of gravity.
4. Like almost any beam balance, the movable weights are connected to the fulcrum point by relatively long lever arms, while the platform on which you stand is connected to the fulcrum point by a very short lever arm. The scale "balances" when the torque provided by your weight (large mass, small lever arm) is equal to that provided by the sliding weights (small mass, large lever arm).
5. (a) If we assume that the pivot point of rotation is the lower left corner of the wall in the picture, then the gravity force acting through the CM provides the torque to keep the wall upright. Note that the gravity force would have a relatively small lever arm (about half the width of the wall). Thus, the sideways force would not have to be particularly large to start to move the wall.
 (b) With the horizontal extension, there are factors that make the wall less likely to overturn:
 - The mass of the second wall is larger, so the torque caused by gravity (helping to keep the wall upright) will be larger for the second wall.
 - The center of gravity of the second wall is farther to the right of the pivot point, so gravity exerts a larger torque to counteract the torque due to \vec{F} .
 - The weight of the ground above the new part of the wall provides a large clockwise torque that helps counteract the torque due to \vec{F} .

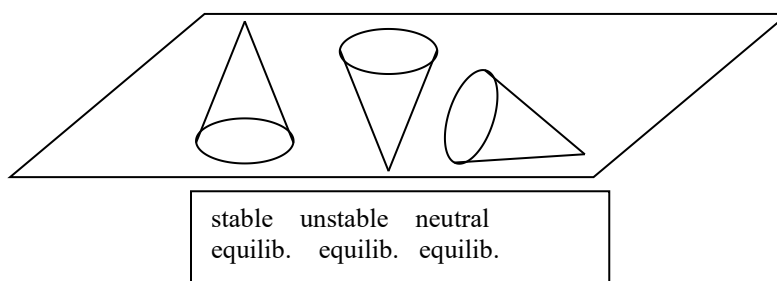
6. If the sum of the forces on an object is not zero, then the CM of the object will accelerate in the direction of the net force. If the sum of the torques on the object is zero, then the object has no angular acceleration. Some examples are:
- A satellite in a circular orbit around the Earth.
 - A block sliding down an inclined plane.
 - An object that is in projectile motion but not rotating.
 - The startup motion of an elevator, changing from rest to having a nonzero velocity.

7. When the person stands near the top, the ladder is more likely to slip. In the accompanying diagram, the force of the person pushing down on the ladder ($M\vec{g}$) causes a clockwise torque about the contact point with the ground, with lever arm d_x . The only force causing a counterclockwise torque about that same point is the reaction force of the wall on the ladder, \vec{F}_W . While the ladder is in equilibrium, \vec{F}_W will be the same magnitude as the frictional force at the ground, \vec{F}_{Gx} . Since \vec{F}_{Gx} has a maximum value, \vec{F}_W will have the same maximum value, and \vec{F}_W will have a maximum counterclockwise torque that it can exert. As the person climbs the ladder, his lever arm gets longer, so the torque due to his weight gets larger.

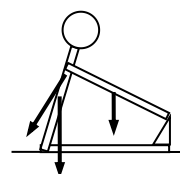


Eventually, if the torque caused by the person is larger than the maximum torque caused by \vec{F}_W , the ladder will start to slip—it will not stay in equilibrium.

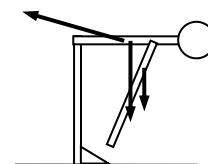
8. The mass of the meter stick is equal to the mass of the rock. Since the meter stick is uniform, its center of mass is at the 50-cm mark. In terms of rotational motion about a pivot at the 25-cm mark, we can treat the stick as though its entire mass is concentrated at the center of mass. The meter stick's mass at the 50-cm mark (25 cm from the pivot) balances the rock at the 0 mark (also 25 cm from the pivot), so the masses must be equal.
9. You lean backward in order to keep your center of mass over your feet. If, due to the heavy load, your center of mass is in front of your feet, you will fall forward.
10. (a) The cone will be in stable equilibrium if it is placed flat on its base. If it is tilted slightly from this position and then released, it will return to the original position.
- (b) The cone will be in unstable equilibrium if it is balanced on its tip. A slight displacement in this case will cause the cone to topple over.
- (c) If the cone is placed on its side, it will be in neutral equilibrium. If the cone is displaced slightly while on its side, it will remain in its new position.



11. When you rise on your tiptoes, your CM shifts forward. Since you are already standing with your nose and abdomen against the door, your CM cannot shift forward. Thus gravity exerts a torque on you and you are unable to stay on your tiptoes—you will return to being flat-footed on the floor.
12. When you start to stand up from a normal sitting position, your CM is not over your point of support (your feet), so gravity will exert a torque about your feet that rotates you back down into the chair. You must lean forward in order that your CM is over your feet so that you can stand up.
13. While you are doing a sit-up, your abdomen muscles provide a torque to rotate you up away from the floor. The force of gravity on your upper half-body tends to pull you back down to the floor, which makes doing sit-ups difficult. The force of gravity on your lower half-body provides a torque that opposes the torque caused by the force of gravity on your upper half-body, making the sit-up a little easier. When your legs are bent, the lever arm for the lower half-body is shorter, so less counter-torque is available.
14. For rotating the upper half-body, the pivot point is near the waist and hips. In that position, the arms have a relatively small torque, even when extended, due to their smaller mass. The more massive trunk-head combination has a very short lever arm, so it also has a relatively small torque. Thus, the force of gravity on the upper body causes relatively little torque about the hips, tending to rotate you forward, and the back muscles need to produce little torque to keep you from rotating forward. The force on the upper half-body due to the back muscles is small, so the (partially rightward) force at the base of the spinal column (not shown in the diagram), to keep the spine in equilibrium, will be small.



When you stand and bend over, the lever arm for the upper body is much larger than while you are sitting, which causes a much larger torque. The CM of the arms is also farther from the support point and causes more torque. The back muscles, assumed to act at the center of the back, do not have a very long lever arm. Thus the back muscles will have to exert a large force to cause a counter-torque that keeps you from falling over. Accordingly, there will have to be a large force (mostly to the right, and not drawn in the diagram) at the base of the spine to keep the spine in equilibrium.



15. Configuration (b) is more likely to be stable. In configuration (a), the CG of the bottom brick is at the edge of the table, and the CG of the top brick is to the right of the edge of the table. Thus the CG of the two-brick system is not above the base of support, and gravity will exert a torque to roll the bricks clockwise off the table. Another way to see this is that more than 50% of the brick mass is not above the base of support—50% of the bottom brick and 75% of the top brick are to the right of the edge of the table. It is not in equilibrium.

In configuration (b), exactly half of the mass (75% of the top brick and 25% of the bottom brick) is over the edge of the table. Thus the CG of the pair is at the edge of the table—it is in unstable equilibrium.

16. A is a point of unstable equilibrium, B is a point of stable equilibrium, and C is a point of neutral equilibrium.
17. The Young's modulus for a bungee cord is much smaller than that for ordinary rope. We know that a bungee cord stretches more easily than ordinary rope. From Eq. 9-4, we have $E = \frac{F/A}{\Delta\ell/\ell_0}$. The value of Young's modulus is inversely proportional to the change in length of a material under a tension. Since the change in length of a bungee cord is much larger than that of an ordinary rope if other conditions are identical (stressing force, unstretched length, cross-sectional area of rope or cord), it must have a smaller Young's modulus.

18. An object under shear stress has equal and opposite forces applied across its opposite faces. This is exactly what happens with a pair of scissors. One blade of the scissors pushes down on the cardboard, while the other blade pushes up with an equal and opposite force, at a slight displacement. This produces a shear stress in the cardboard, which causes it to fail.
19. Concrete or stone should definitely *not* be used for the support on the left. The left-hand support pulls downward on the beam, so the beam must pull upward on the support. Therefore, the support will be under tension and should not be made of ordinary concrete or stone, since these materials are weak under tension. The right-hand support pushes up on the beam, so the beam pushes down on it; it will therefore be under a compression force. Making this support of concrete or stone would be acceptable.

Responses to MisConceptual Questions

1. (d) In attempting to solve this problem, students frequently try to divide the beam into multiple parts to calculate the torque due to the weight of the beam. The beam should be considered as a single object with its weight acting at its center of mass ($\frac{1}{4}\ell$ from the pivot). Since the woman is on the opposite side of the pivot and at the same distance as the beam's center of mass, their forces of gravity and masses must be equal.
2. (d) A common misconception is that a nonrotating object has an axis of rotation. If an object is not rotating, it is not rotating about any arbitrary point. When solving an equilibrium problem with no rotation, the student can select any axis for the torques that facilitates solving the problem.
3. (a) Students might think that for the net force on the beam to be zero, the tension would equal the weight of the beam. However, this does not take into account the force that the wall exerts on the hinged end. Students might assume that the tension is equal to half of the beam's weight. However, this does not take into account the vector nature of the tension. The vertical component of the tension is equal to half of the weight, but there is also a horizontal component. Adding these two components yields a tension at least half of the weight of the beam.
4. (c) Drawing a free-body diagram for this problem will resolve student misconceptions. When the ball is pulled to the side, there are three forces acting on the ball: the vertical weight, the horizontal applied force, and the tension along the direction of the cable. Resolving the tension into horizontal and vertical parts and applying Newton's second law in equilibrium, we can see that the applied force is equal to the horizontal component of the tension.
5. (a) As the child leans forward, her center of mass moves closer to the pivot point, which decreases her lever arm. The seesaw is no longer in equilibrium. Since the torque on her side has decreased, she will rise.
6. (c) A common misconception is that each cord will support one-half of the weight regardless of the angle. An analysis of the forces using Newton's second law in equilibrium shows that the horizontal components of the tension are equal. Since cord A makes a larger angle with the horizontal, it has a greater total tension and therefore supports more than half the suspended weight.
7. (c) The applied force is proportional to the stress, so increasing the force will affect the stress. The strain is how the rope responds to the stress. Increasing the force will then affect the strain. Young's modulus is the constant of proportionality between the stress and strain. It is determined by the properties of the material, so it is not affected by pulling on the rope.

8. (e) Students may consider the tension equal to the woman's weight, or half of the woman's weight, if they do not consider the vector nature of the forces. A free-body diagram for the point at the bottom of the woman's foot shows three forces acting: the weight of the woman and the diagonal tensions in the wire on each side of her foot. Applying Newton's second law in equilibrium in the vertical direction shows that the vertical component of the tension must equal half of her weight. Since vertical displacement is small compared to the horizontal length of the wire, the total tension is much greater than the vertical component of the tension.
9. (d) When the length, width, and number of floors are doubled, the weight of the garage increases by a factor of eight. To keep the stress on the columns unchanged, the area of the columns should also increase by a factor of eight.
10. (d) The stress (applied force) is proportional to the strain (change in length). Doubling the stress will cause the strain to double also.

Solutions to Problems

1. If the tree is not accelerating, then the net force in all directions is 0.

$$\sum F_x = F_A + F_B \cos 105^\circ + F_{Cx} = 0 \rightarrow$$

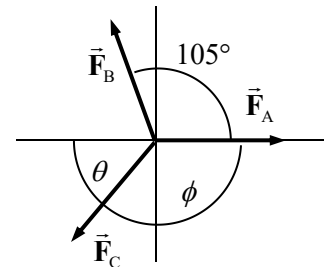
$$F_{Cx}} = -F_A - F_B \cos 105^\circ = -385 \text{ N} - (475 \text{ N}) \cos 105^\circ = -262.1 \text{ N}$$

$$\sum F_y = F_B \sin 105^\circ + F_{Cy} = 0 \rightarrow$$

$$F_{Cy} = -F_B \sin 105^\circ = -(475 \text{ N}) \sin 105^\circ = -458.8 \text{ N}$$

$$F_C = \sqrt{F_{Cx}^2 + F_{Cy}^2} = \sqrt{(-262.1 \text{ N})^2 + (-458.8 \text{ N})^2} = 528.4 \text{ N} \approx \boxed{528 \text{ N}}$$

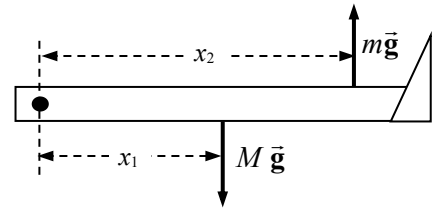
$$\theta = \tan^{-1} \frac{F_{Cy}}{F_{Cx}} = \tan^{-1} \frac{-458.8 \text{ N}}{-262.1 \text{ N}} = 60.3^\circ, \phi = 180^\circ - 60.3^\circ = \boxed{120^\circ}$$



So \vec{F}_C is 528 N, at an angle of 120° clockwise from \vec{F}_A . The angle has 3 significant figures.

2. Because the mass m is stationary, the tension in the rope pulling up on the sling must be mg , and the force of the sling on the leg must be mg , upward. Calculate torques about the hip joint, with counterclockwise torque taken as positive. See the free-body diagram for the leg. Note that the forces on the leg exerted by the hip joint are not drawn, because they do not exert a torque about the hip joint.

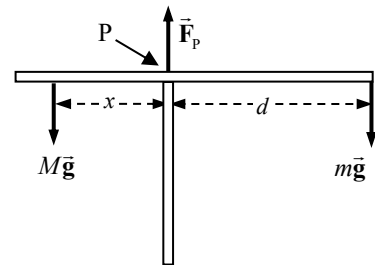
$$\sum \tau = mgx_2 - Mgx_1 = 0 \rightarrow m = M \frac{x_1}{x_2} = (15.0 \text{ kg}) \frac{(35.0 \text{ cm})}{(78.0 \text{ cm})} = \boxed{6.73 \text{ kg}}$$



3. (a) See the free-body diagram. Calculate torques about the pivot point P labeled in the diagram. The upward force at the pivot will not have any torque. The total torque is zero, since the crane is in equilibrium.

$$\sum \tau = Mgx - mgd = 0 \rightarrow$$

$$x = \frac{md}{M} = \frac{(2800 \text{ kg})(7.7 \text{ m})}{(9500 \text{ kg})} = \boxed{2.3 \text{ m}}$$



- (b) Again, we sum torques about the pivot point. Mass m is the unknown in this case, and the counterweight is at its maximum distance from the pivot.

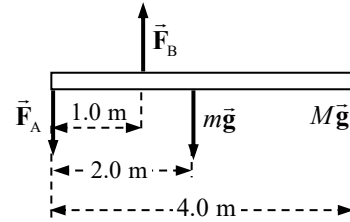
$$\sum \tau = Mg x_{\max} - m_{\max} g d = 0 \rightarrow m_{\max} = \frac{M x_{\max}}{d} = \frac{(9500 \text{ kg})(3.4 \text{ m})}{(7.7 \text{ kg})} = \boxed{4200 \text{ kg}}$$

4. Her torque is her weight times the distance x between the diver and the left support post.

$$\tau = mgx \rightarrow m = \frac{\tau}{gx} = \frac{1800 \text{ m} \cdot \text{N}}{(9.80 \text{ m/s}^2)(4.0 \text{ m})} = \boxed{46 \text{ kg}}$$

5. (a) Let $m = 0$. Calculate the net torque about the left end of the diving board, with counterclockwise torques positive. Since the board is in equilibrium, the net torque is zero.

$$\begin{aligned} \sum \tau &= F_B(1.0 \text{ m}) - Mg(4.0 \text{ m}) = 0 \rightarrow \\ F_B &= 4Mg = 4(52 \text{ kg})(9.80 \text{ m/s}^2) = 2038 \text{ N} \\ &\approx \boxed{2.0 \times 10^3 \text{ N, up}} \end{aligned}$$



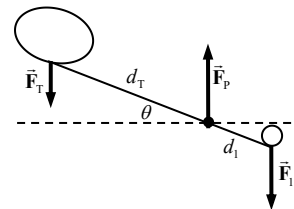
Use Newton's second law in the vertical direction to find F_A .

$$\begin{aligned} \sum F_y &= F_B - Mg - F_A = 0 \rightarrow \\ F_A &= F_B - Mg = 4Mg - Mg = 3Mg = 3(52 \text{ kg})(9.80 \text{ m/s}^2) = 1529 \text{ N} \approx \boxed{1500 \text{ N, down}} \end{aligned}$$

- (b) Repeat the basic process, but with $m = 28 \text{ kg}$. The weight of the board will add more clockwise torque.

$$\begin{aligned} \sum \tau &= F_B(1.0 \text{ m}) - mg(2.0 \text{ m}) - Mg(4.0 \text{ m}) = 0 \rightarrow \\ F_B &= 4Mg + 2mg = [4(52 \text{ kg}) + 2(28 \text{ kg})](9.80 \text{ m/s}^2) = 2587 \text{ N} \approx \boxed{2600 \text{ N, up}} \\ \sum F_y &= F_B - Mg - mg - F_A = 0 \rightarrow \\ F_A &= F_B - Mg - mg = 4Mg + 2mg - Mg - mg = 3Mg + mg \\ &= [3(52 \text{ kg}) + 28 \text{ kg}](9.80 \text{ m/s}^2) = 1803 \text{ N} \approx \boxed{1800 \text{ N, down}} \end{aligned}$$

6. Since each half of the forceps is in equilibrium, the net torque on each half of the forceps is zero. Calculate torques with respect to an axis perpendicular to the plane of the forceps, through point P, counterclockwise being positive. Consider a force diagram for one-half of the forceps. \vec{F}_1 is the force on the half-forceps due to the plastic rod, and force \vec{F}_p is the force on the half-forceps from the pin joint. \vec{F}_p exerts no torque about point P.



$$\sum \tau = F_T d_T \cos \theta - F_1 d_1 \cos \theta = 0 \rightarrow F_1 = F_T \frac{d_T}{d_1} = (11.0 \text{ N}) \frac{8.50 \text{ cm}}{2.70 \text{ cm}} = 34.6 \text{ N}$$

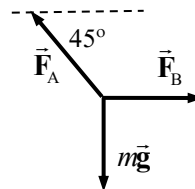
The force that the forceps exerts on the rod is the opposite of \vec{F}_1 , so it is also $\boxed{34.6 \text{ N}}$.

7. Write Newton's second law for the junction, in both the x and y directions.

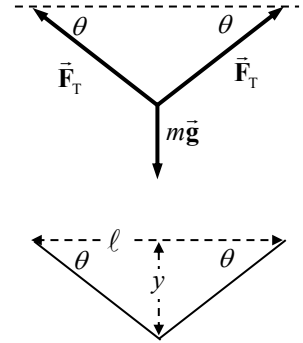
$$\sum F_x = F_B - F_A \cos 45^\circ = 0$$

From this, we see that $F_A > F_B$. Thus set $F_A = 1660 \text{ N}$.

$$\begin{aligned} \sum F_y &= F_A \sin 45^\circ - mg = 0 \\ mg &= F_A \sin 45^\circ = (1660 \text{ N}) \sin 45^\circ = 1174 \text{ N} \approx \boxed{1200 \text{ N}} \end{aligned}$$



8. Since the backpack is midway between the two trees, the angles in the diagram are equal. Write Newton's second law for the vertical direction for the point at which the backpack is attached to the cord, with the weight of the backpack being the downward vertical force. The angle is determined by the distance between the trees and the amount of sag at the midpoint, as illustrated in the second diagram.



$$(a) \quad \theta = \tan^{-1} \frac{y}{\ell/2} = \tan^{-1} \frac{1.5 \text{ m}}{3.3 \text{ m}} = 24.4^\circ$$

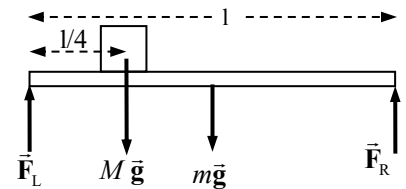
$$\sum F_y = 2F_T \sin \theta_1 - mg = 0 \rightarrow$$

$$F_T = \frac{mg}{2 \sin \theta_1} = \frac{(19 \text{ kg})(9.80 \text{ m/s}^2)}{2 \sin 24.4^\circ} = 225.4 \text{ N} \approx \boxed{230 \text{ N}}$$

$$(b) \quad \theta = \tan^{-1} \frac{y}{\ell/2} = \tan^{-1} \frac{0.15 \text{ m}}{3.3 \text{ m}} = 2.60^\circ$$

$$F_T = \frac{mg}{2 \sin \theta_1} = \frac{(19 \text{ kg})(9.80 \text{ m/s}^2)}{2 \sin 2.60^\circ} = 2052 \text{ N} \approx \boxed{2100 \text{ N}}$$

9. Let m be the mass of the beam, and M be the mass of the piano. Calculate torques about the left end of the beam, with counterclockwise torques positive. The conditions of equilibrium for the beam are used to find the forces that the support exerts on the beam.



$$\sum \tau = F_R \ell - mg \left(\frac{1}{2} \ell \right) - Mg \left(\frac{1}{4} \ell \right) = 0$$

$$F_R = \left(\frac{1}{2} m + \frac{1}{4} M \right) g = \left[\frac{1}{2} (110 \text{ kg}) + \frac{1}{4} (320 \text{ kg}) \right] (9.80 \text{ m/s}^2) = 1320 \text{ N}$$

$$\sum F_y = F_L + F_R - mg - Mg = 0$$

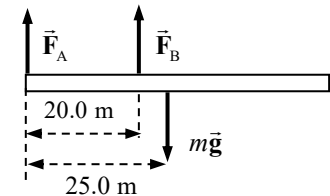
$$F_L = (m + M)g - F_R = (430 \text{ kg})(9.80 \text{ m/s}^2) - 1.32 \times 10^3 \text{ N} = 2890 \text{ N}$$

The forces on the supports are equal in magnitude and opposite in direction to the above two results.

$$\boxed{F_R = 1300 \text{ N down}}$$

$$\boxed{F_L = 2900 \text{ N down}}$$

10. Calculate torques about the left end of the beam, with counterclockwise torques positive. The conditions of equilibrium for the beam are used to find the forces that the support exerts on the beam.



$$\sum \tau = F_B (20.0 \text{ m}) - mg (25.0 \text{ m}) = 0 \rightarrow$$

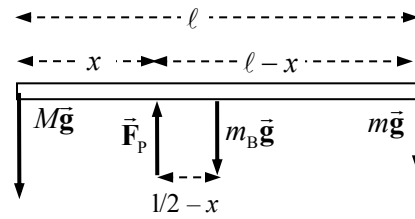
$$F_B = \frac{25.0}{20.0} mg = (1.25)(1200 \text{ kg})(9.80 \text{ m/s}^2) = \boxed{1.5 \times 10^4 \text{ N}}$$

$$\sum F_y = F_A + F_B - mg = 0$$

$$F_A = mg - F_B = mg - 1.25mg = -0.25mg = -(0.25)(1200 \text{ kg})(9.80 \text{ m/s}^2) = \boxed{-2900 \text{ N}}$$

Notice that \vec{F}_A points down.

11. The pivot should be placed so that the net torque on the board is zero. We calculate torques about the pivot point, with counterclockwise torques positive. The upward force \vec{F}_p at the pivot point is shown, but it exerts no torque about the pivot point. The mass of the child is m , the mass of the adult is M , the mass of the board is m_B , and the center of gravity is at the middle of the board.



- (a) Ignore the force $m_B g$.

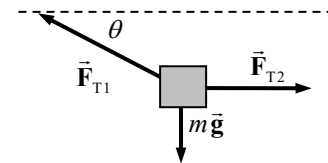
$$\sum \tau = Mg x - mg(\ell - x) = 0 \rightarrow x = \frac{m}{m+M} \ell = \frac{(25 \text{ kg})}{(25 \text{ kg} + 75 \text{ kg})} (9.0 \text{ m}) = 2.25 \text{ m} \approx \boxed{2.3 \text{ m from adult}}$$

- (b) Include the force $m_B g$.

$$\sum \tau = Mg x - mg(\ell - x) - m_B g(\ell/2 - x) = 0$$

$$x = \frac{(m + m_B/2)}{(M + m + m_B)} \ell = \frac{(25 \text{ kg} + 7.5 \text{ kg})}{(75 \text{ kg} + 25 \text{ kg} + 15 \text{ kg})} (9.0 \text{ m}) = 2.54 \text{ m} \approx \boxed{2.5 \text{ m from adult}}$$

12. Using the free-body diagram, write Newton's second law for both the horizontal and vertical directions, with net forces of zero.



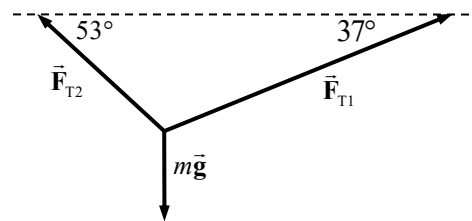
$$\sum F_x = F_{T2} - F_{T1} \cos \theta = 0 \rightarrow F_{T2} = F_{T1} \cos \theta$$

$$\sum F_y = F_{T1} \sin \theta - mg = 0 \rightarrow F_{T1} = \frac{mg}{\sin \theta}$$

$$F_{T2} = F_{T1} \cos \theta = \frac{mg}{\sin \theta} \cos \theta = \frac{mg}{\tan \theta} = \frac{(190 \text{ kg})(9.80 \text{ m/s}^2)}{\tan 33^\circ} = 2867 \text{ N} \approx \boxed{2900 \text{ N}}$$

$$F_{T1} = \frac{mg}{\sin \theta} = \frac{(190 \text{ kg})(9.80 \text{ m/s}^2)}{\sin 33^\circ} = 3418 \text{ N} \approx \boxed{3400 \text{ N}}$$

13. Draw a free-body diagram of the junction of the three wires. The tensions can be found from the conditions for force equilibrium.



$$\sum F_x = F_{T1} \cos 37^\circ - F_{T2} \cos 53^\circ = 0 \rightarrow F_{T2} = \frac{\cos 37^\circ}{\cos 53^\circ} F_{T1}$$

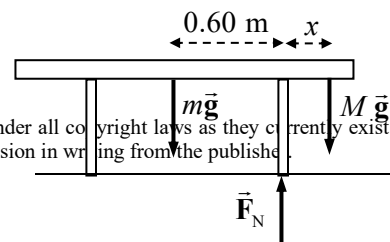
$$\sum F_y = F_{T1} \sin 37^\circ + F_{T2} \sin 53^\circ - mg = 0$$

$$F_{T1} \sin 37^\circ + \frac{\cos 37^\circ}{\cos 53^\circ} F_{T1} \sin 53^\circ - mg = 0 \rightarrow$$

$$F_{T1} = \frac{(33 \text{ kg})(9.80 \text{ m/s}^2)}{\sin 37^\circ + \frac{\cos 37^\circ}{\cos 53^\circ} \sin 53^\circ} = 194.6 \text{ N} \approx \boxed{190 \text{ N}}$$

$$F_{T2} = \frac{\cos 37^\circ}{\cos 53^\circ} F_{T1} = \frac{\cos 37^\circ}{\cos 53^\circ} (1.946 \times 10^2 \text{ N}) = 258.3 \text{ N} \approx \boxed{260 \text{ N}}$$

14. The table is symmetric, so the person can sit near either edge



and the same distance will result. We assume that the person (mass M) is on the right side of the table and that the table

(mass m) is on the verge of tipping, so that the left leg is on the verge of lifting off the floor. There will then be no normal force between the left leg of the table and the floor. Calculate torques about the right leg of the table such that the normal force between the table and the floor causes no torque.

Counterclockwise torques are taken to be positive. The conditions of equilibrium for the table are used to find the person's location.

$$\sum \tau = mg(0.60 \text{ m}) - Mgx = 0 \rightarrow x = (0.60 \text{ m}) \frac{m}{M} = (0.60 \text{ m}) \frac{24.0 \text{ kg}}{66.0 \text{ kg}} = 0.218 \text{ m}$$

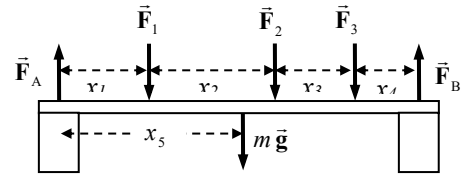
Thus the distance from the edge of the table is $0.50 \text{ m} - 0.218 \text{ m} = \boxed{0.28 \text{ m}}$.

15. The bottle opener will pull upward on the cork with a force of magnitude F_{cork} , so there is a downward force on the opener of magnitude F_{cork} . We assume that there is no net torque on the opener, so it has no angular acceleration. Calculate torques about the rim of the bottle where the opener is resting on the rim.

$$\sum \tau = F(79 \text{ mm}) - F_{\text{cork}}(9 \text{ mm}) = 0 \rightarrow$$

$$F = \frac{9}{70} F_{\text{cork}} = \frac{9}{79} (200 \text{ N}) \text{ to } \frac{9}{79} (400 \text{ N}) = 22.8 \text{ N to } 45.6 \text{ N} \approx \boxed{20 \text{ N to } 50 \text{ N}}$$

16. The beam is in equilibrium, so both the net torque and net force on it must be zero. From the free-body diagram, calculate the net torque about the center of the left support, with counterclockwise torques as positive. Calculate the net force, with upward as positive. Use those two equations to find F_A and F_B .



$$\sum \tau = F_B(x_1 + x_2 + x_3 + x_4) - F_1x_1 - F_2(x_1 + x_2) - F_3(x_1 + x_2 + x_3) - mgx_5$$

$$F_B = \frac{F_1x_1 + F_2(x_1 + x_2) + F_3(x_1 + x_2 + x_3) + mgx_5}{(x_1 + x_2 + x_3 + x_4)}$$

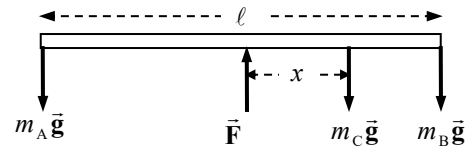
$$= \frac{(4300 \text{ N})(2.0 \text{ m}) + (3100 \text{ N})(6.0 \text{ m}) + (2200 \text{ N})(9.0 \text{ m}) + (280 \text{ kg})(9.80 \text{ m/s}^2)(5.0 \text{ m})}{10.0 \text{ m}}$$

$$= 6072 \text{ N} \approx \boxed{6100 \text{ N}}$$

$$\sum F = F_A + F_B - F_1 - F_2 - F_3 - mg = 0$$

$$F_A = F_1 + F_2 + F_3 + mg - F_B = 9600 \text{ N} + (280 \text{ kg})(9.80 \text{ m/s}^2) - 6072 \text{ N} = 6272 \text{ N} \approx \boxed{6300 \text{ N}}$$

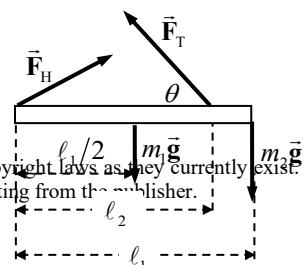
17. From the free-body diagram, the conditions of equilibrium are used to find the location of the girl (mass m_C). The 45-kg boy is represented by m_A and the 35-kg boy by m_B . Calculate torques about the center of the seesaw, and take counterclockwise torques to be positive. The upward force of the fulcrum on the seesaw (\vec{F}) causes no torque about the center.



$$\sum \tau = m_A g \left(\frac{1}{2} \ell \right) - m_C g x - m_B g \left(\frac{1}{2} \ell \right) = 0$$

$$x = \frac{(m_A - m_B)}{m_C} \left(\frac{1}{2} \ell \right) = \frac{(45 \text{ kg} - 35 \text{ kg})}{25 \text{ kg}} \frac{1}{2} (3.2 \text{ m}) = \boxed{0.64 \text{ m}}$$

18. The beam is in equilibrium. Use the conditions of equilibrium to calculate



the tension in the wire and the forces at the hinge. Calculate torques about the hinge, and take counterclockwise torques to be positive.

$$\begin{aligned}\sum \tau &= (F_T \sin \theta) \ell_2 - m_1 g \ell_1 / 2 - m_2 g \ell_1 = 0 \rightarrow \\ F_T &= \frac{\frac{1}{2} m_1 g \ell_1 + m_2 g \ell_1}{\ell_2 \sin \theta} = \frac{\frac{1}{2} (155 \text{ N})(1.70 \text{ m}) + (215 \text{ N})(1.70 \text{ m})}{(1.35 \text{ m})(\sin 35.0^\circ)} \\ &= 642.2 \text{ N} \approx \boxed{642 \text{ N}}\end{aligned}$$

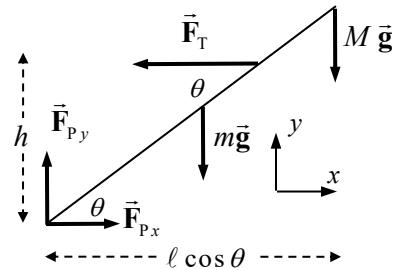
$$\sum F_x = F_{Hx} - F_T \cos \theta = 0 \rightarrow F_{Hx} = F_T \cos \theta = (642.2 \text{ N}) \cos 35.0^\circ = 526.1 \text{ N} \approx \boxed{526 \text{ N}}$$

$$\sum F_y = F_{Hy} + F_T \sin \theta - m_1 g - m_2 g = 0 \rightarrow$$

$$F_{Hy} = m_1 g + m_2 g - F_T \sin \theta = 155 \text{ N} + 215 \text{ N} - (642.2 \text{ N}) \sin 35.0^\circ = 1.649 \text{ N} \approx \boxed{2 \text{ N}}$$

19. (a) The pole is in equilibrium, so the net torque on it must be zero. From the free-body diagram, calculate the net torque about the lower end of the pole, with counterclockwise torques as positive. Use that calculation to find the tension in the cable. The length of the pole is ℓ .

$$\begin{aligned}\sum \tau &= F_T h - mg(\ell/2) \cos \theta - Mg \ell \cos \theta = 0 \\ F_T &= \frac{(m/2 + M) g \ell \cos \theta}{h} \\ &= \frac{(6.0 \text{ kg} + 21.5 \text{ kg})(9.80 \text{ m/s}^2)(7.20 \text{ m}) \cos 37^\circ}{3.80 \text{ m}} = 407.8 \text{ N} \approx \boxed{410 \text{ N}}\end{aligned}$$



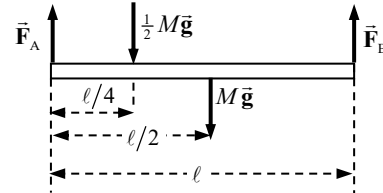
- (b) The net force on the pole is also zero since it is in equilibrium. Write Newton's second law in both the x and y directions to solve for the forces at the pivot.

$$\sum F_x = F_{Px} - F_T = 0 \rightarrow F_{Px} = F_T = \boxed{410 \text{ N}}$$

$$\sum F_y = F_{Py} - mg - Mg = 0 \rightarrow F_{Py} = (m + M)g = (33.5 \text{ kg})(9.80 \text{ m/s}^2) = \boxed{328 \text{ N}}$$

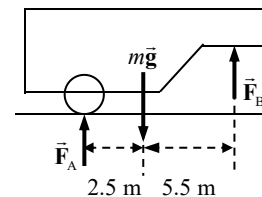
20. The center of gravity of each beam is at its geometric center. Calculate torques about the left end of the beam, and take counterclockwise torques to be positive. The conditions of equilibrium for the beam are used to find the forces that the support exerts on the beam.

$$\begin{aligned}\sum \tau &= F_B \ell - Mg(\ell/2) - \frac{1}{2} Mg(\ell/4) = 0 \rightarrow \\ F_B &= \frac{5}{8} Mg = \frac{5}{8} (940 \text{ kg})(9.80 \text{ m/s}^2) = 5758 \text{ N} \approx \boxed{5800 \text{ N}} \\ \sum F_y &= F_A + F_B - Mg - \frac{1}{2} Mg = 0 \rightarrow \\ F_A &= \frac{3}{2} Mg - F_B = \frac{7}{8} Mg = \frac{7}{8} (940 \text{ kg})(9.80 \text{ m/s}^2) = 8061 \text{ N} \approx \boxed{8100 \text{ N}}\end{aligned}$$



21. To find the normal force exerted on the road by the trailer tires, take the torques about point B, with counterclockwise torques as positive.

$$\begin{aligned}\sum \tau &= mg(5.5 \text{ m}) - F_A(8.0 \text{ m}) = 0 \rightarrow \\ F_A &= mg \left(\frac{5.5 \text{ m}}{8.0 \text{ m}} \right) = (2500 \text{ kg})(9.80 \text{ m/s}^2) \left(\frac{5.5 \text{ m}}{8.0 \text{ m}} \right) = 16,844 \text{ N} \\ &\approx \boxed{1.7 \times 10^4 \text{ N}}\end{aligned}$$

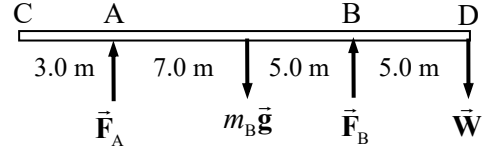


The net force in the vertical direction must be zero.

$$\sum F_y = F_B + F_A - mg = 0 \rightarrow$$

$$F_B = mg - F_A = (2500 \text{ kg})(9.80 \text{ m/s}^2) - 16,844 \text{ N} = 7656 \text{ N} \approx \boxed{7.7 \times 10^3 \text{ N}}$$

22. (a) For the extreme case of the beam being ready to tip, there would be no normal force at point A from the support. Use the free-body diagram to write the equation of rotational equilibrium under that condition to find the weight of the person, with $F_A = 0$. Take torques about the location of



support B, and call counterclockwise torques positive. \vec{W} is the weight of the person, and m_B is the mass of the beam.

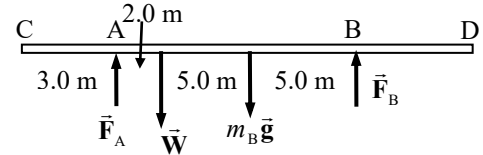
$$\sum \tau = m_B g(5.0 \text{ m}) - W(5.0 \text{ m}) = 0 \rightarrow$$

$$W = m_B g = \boxed{650 \text{ N}}$$

- (b) With the person standing at point D, we have already assumed that $F_A = 0$. The net force in the vertical direction must also be zero.

$$\sum F_y = F_A + F_B - m_B g - W = 0 \rightarrow F_B = m_B g + W = 650 \text{ N} + 650 \text{ N} = \boxed{1.30 \times 10^3 \text{ N}}$$

- (c) The person moves to a different spot, so the free-body diagram changes again as shown. Again use the net torque about support B and then use the net vertical force.

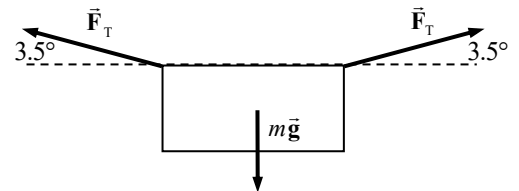


$$\sum \tau = m_B g(5.0 \text{ m}) + W(10.0 \text{ m}) - F_A(12.0 \text{ m}) = 0$$

$$F_A = \frac{m_B g(5.0 \text{ m}) + W(10.0 \text{ m})}{12.0 \text{ m}} = \frac{(650 \text{ N})(5.0 \text{ m}) + (650 \text{ N})(10.0 \text{ m})}{12.0 \text{ m}} = \boxed{810 \text{ N}}$$

$$\sum F_y = F_A + F_B - m_B g - W = 0 \rightarrow F_B = m_B g + W - F_A = 1300 \text{ N} - 810 \text{ N} = \boxed{490 \text{ N}}$$

23. Draw the free-body diagram for the sheet, and write Newton's second law for the vertical direction. Note that the tension is the same in both parts of the clothesline.



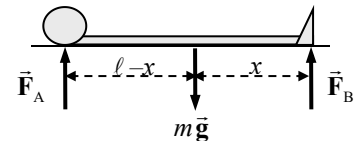
$$\sum F_y = F_T \sin 3.5^\circ + F_T \sin 3.5^\circ - mg = 0 \rightarrow$$

$$F_T = \frac{mg}{2(\sin 3.5^\circ)} = \frac{(0.75 \text{ kg})(9.80 \text{ m/s}^2)}{2(\sin 3.5^\circ)}$$

$$= \boxed{60 \text{ N}} \text{ (2 significant figures)}$$

The 60-N tension is much higher than the ~ 7.5 -N weight of the sheet because of the small angle. Only the vertical components of the tension are supporting the sheet. Since the angle is small, the tension has to be large to have a large enough vertical component to hold up the sheet.

24. The person is in equilibrium, so both the net torque and net force must be zero. From the free-body diagram, calculate the net torque about the center of gravity, with counterclockwise torques as positive. Use that calculation to find the location of the center of gravity, a distance x from the feet.



$$\sum \tau = F_B x - F_A (\ell - x) = 0$$

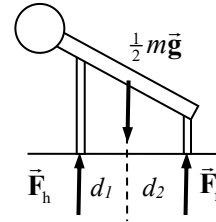
$$x = \frac{F_A}{F_A + F_B} \ell = \frac{m_A g}{m_A g + m_B g} \ell = \frac{m_A}{m_A + m_B} \ell = \frac{35.1 \text{ kg}}{31.6 \text{ kg} + 35.1 \text{ kg}} (1.72 \text{ m}) = \boxed{9.05 \times 10^{-1} \text{ m}}$$

The center of gravity is about 90.5 cm from the feet.

25. (a) The man is in equilibrium, so the net force and the net torque on him must be zero. We use half of his weight and then consider the force just on one hand and one foot, assuming that he is symmetrical. Take torques about the point where the foot touches the ground, with counterclockwise as positive.

$$\sum \tau = \frac{1}{2} mg d_2 - F_h (d_1 + d_2) = 0$$

$$F_h = \frac{mg d_2}{2(d_1 + d_2)} = \frac{(68 \text{ kg})(9.80 \text{ m/s}^2)(0.95 \text{ m})}{2(1.37 \text{ m})} = 231 \text{ N} \approx \boxed{230 \text{ N}}$$



- (b) Use Newton's second law for vertical forces to find the force on the feet.

$$\sum F_y = 2F_h + 2F_f - mg = 0$$

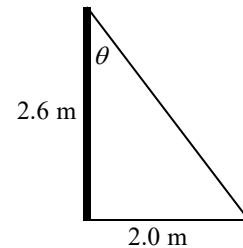
$$F_f = \frac{1}{2} mg - F_h = \frac{1}{2} (68 \text{ kg})(9.80 \text{ m/s}^2) - 231 \text{ N} = 103 \text{ N} \approx \boxed{100 \text{ N}}$$

The value of 100 N has 2 significant figures.

26. First consider the triangle made by the pole and one of the wires (first diagram). It has a vertical leg of 2.6 m and a horizontal leg of 2.0 m. The angle that the tension (along the wire) makes with the vertical is

$$\theta = \tan^{-1} \frac{2.0}{2.6} = 37.6^\circ. \text{ The part of the tension that is parallel to the ground is}$$

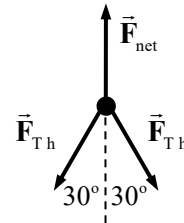
$$\text{therefore } F_{T_h} = F_T \sin \theta.$$



Now consider a top view of the pole, showing only force parallel to the ground (second diagram). The horizontal parts of the tension lie as the sides of an equilateral triangle, so each makes a 30° angle with the tension force of the net. Write the equilibrium equation for the forces along the direction of the tension in the net.

$$\sum F = F_{\text{net}} - 2F_{T_h} \cos 30^\circ = 0 \rightarrow$$

$$F_{\text{net}} = 2F_T \sin \theta \cos 30^\circ = 2(115 \text{ N}) \sin 37.6^\circ \cos 30^\circ = 121.5 \text{ N} \approx \boxed{120 \text{ N}}$$



27. (a) Choose the coordinates as shown in the free-body diagram.
(b) Write the equilibrium conditions for the horizontal and vertical forces.

$$\sum F_x = F_{\text{rope}} \sin \phi - F_{\text{hinge}}^{\text{horiz}} = 0 \rightarrow$$

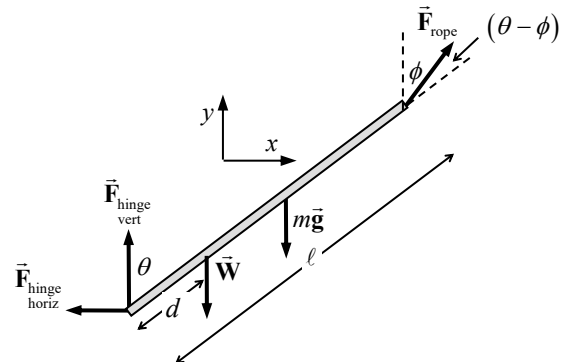
$$F_{\text{hinge}}^{\text{horiz}} = F_{\text{rope}} \sin \phi = (85 \text{ N}) \sin 37^\circ = \boxed{51 \text{ N}}$$

$$\sum F_y = F_{\text{rope}} \cos \phi + F_{\text{hinge}}^{\text{vert}} - mg - W = 0 \rightarrow$$

$$F_{\text{hinge}}^{\text{vert}} = mg + W - F_{\text{rope}} \cos \phi = (3.8 \text{ kg})(9.80 \text{ m/s}^2)$$

$$+22 \text{ N} - (85 \text{ N}) \cos 37^\circ = -8.6 \text{ N} \approx \boxed{-9 \text{ N}}$$

So the vertical hinge force actually points downward.



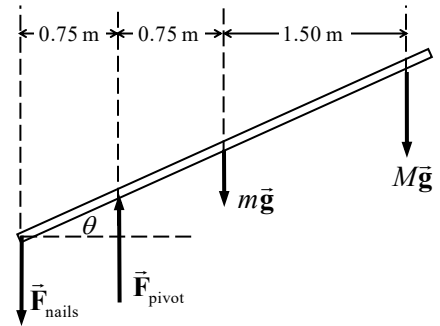
- (c) We take torques about the hinge point, with clockwise torques as positive.

$$\sum \tau = Wd \sin \theta + mg \left(\frac{1}{2} \ell \right) \sin \theta - F_{\text{rope}} \ell \sin (\theta - \phi) = 0 \rightarrow$$

$$d = \frac{F_{\text{rope}} \ell \sin (\theta - \phi) - mg \left(\frac{1}{2} \ell \right) \sin \theta}{W \sin \theta}$$

$$= \frac{(85 \text{ N})(5.0 \text{ m}) \sin 16^\circ - (3.8 \text{ kg})(9.80 \text{ m/s}^2)(2.5 \text{ m}) \sin 53^\circ}{(22 \text{ N}) \sin 53^\circ} = 2.436 \text{ m} \approx \boxed{2.4 \text{ m}}$$

28. See the free-body diagram. Take torques about the pivot point with clockwise torques as positive. The plank is in equilibrium. Let m represent the mass of the plank and M represent the mass of the person. The minimum nail force would occur if there was no normal force pushing up on the left end of the board.

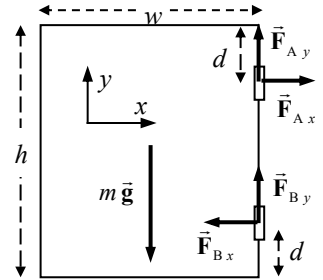


$$\sum \tau = mg(0.75 \text{ m}) \cos \theta + Mg(2.25 \text{ m}) \cos \theta - F_{\text{nails}}(0.75 \text{ m}) \cos \theta = 0 \rightarrow$$

$$F_{\text{nails}} = \frac{mg(0.75 \text{ m}) + Mg(2.25 \text{ m})}{(0.75 \text{ m})} = mg + 3Mg$$

$$= (45 \text{ kg} + 3(65 \text{ kg}))(9.80 \text{ m/s}^2) = 2352 \text{ N} \approx \boxed{2400 \text{ N}}$$

29. The forces on the door are due to gravity and the hinges. Since the door is in equilibrium, the net torque and net force must be zero. Write the three equations of equilibrium. Calculate torques about the bottom hinge, with counterclockwise torques as positive. From the statement of the problem, $F_{Ay} = F_{By} = \frac{1}{2}mg$.



$$\sum \tau = mg \frac{w}{2} - F_{Ax}(h - 2d) = 0$$

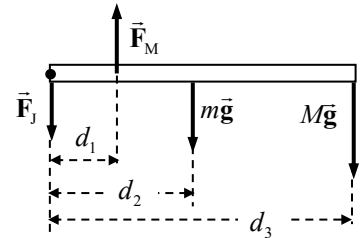
$$F_{Ax} = \frac{mgw}{2(h - 2d)} = \frac{(13.0 \text{ kg})(9.80 \text{ m/s}^2)(1.30 \text{ m})}{2(2.30 \text{ m} - 0.80 \text{ m})} = \boxed{55.2 \text{ N}}$$

$$\sum F_x = F_{Ax} - F_{Bx} = 0 \rightarrow F_{Bx} = F_{Ax} = \boxed{55.2 \text{ N}}$$

$$\sum F_y = F_{Ay} + F_{By} - mg = 0 \rightarrow F_{Ay} = F_{By} = \frac{1}{2}mg = \frac{1}{2}(13.0 \text{ kg})(9.80 \text{ m/s}^2) = \boxed{63.7 \text{ N}}$$

30. The arm is in equilibrium. Take torques about the elbow joint (the dot in the free-body diagram), so that the force at the elbow joint does not enter the calculation. Counterclockwise torques are positive. The mass of the lower arm is $m = 2.0 \text{ kg}$, and the mass of the load is M .

It is given that $F_M = 450 \text{ N}$.



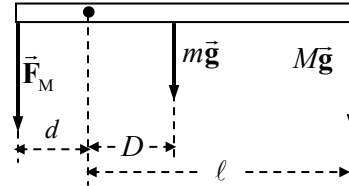
$$\sum \tau = F_M d_1 - mgd_2 - Mg d_3 = 0 \rightarrow$$

$$M = \frac{F_M d_1 - mgd_2}{gd_3} = \frac{(450 \text{ N})(0.060 \text{ m}) - (2.0 \text{ kg})(9.80 \text{ m/s}^2)(0.15 \text{ m})}{(9.80 \text{ m/s}^2)(0.35 \text{ m})} = \boxed{7.0 \text{ kg}}$$

31. Calculate the torques about the elbow joint (the dot in the free-body diagram). The arm is in equilibrium. Counterclockwise torques are positive.

$$\sum \tau = F_M d - mgD - Mg \ell = 0$$

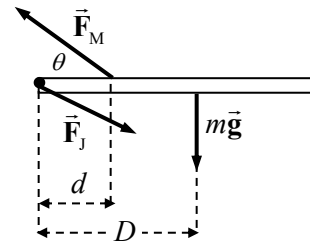
$$F_M = \frac{mD + M \ell}{d} g = \left[\frac{(2.3 \text{ kg})(0.12 \text{ m}) + (7.3 \text{ kg})(0.300 \text{ m})}{0.025 \text{ m}} \right] (9.80 \text{ m/s}^2) = \boxed{970 \text{ N}}$$



32. (a) Calculate the torques about the elbow joint (the dot in the free-body diagram). The arm is in equilibrium. Take counterclockwise torques as positive.

$$\sum \tau = (F_M \sin \theta) d - mgD = 0 \rightarrow$$

$$F_M = \frac{mgD}{d \sin \theta} = \frac{(3.3 \text{ kg})(9.80 \text{ m/s}^2)(0.24 \text{ m})}{(0.12 \text{ m}) \sin 15^\circ} = 249.9 \text{ N} \approx \boxed{250 \text{ N}}$$



- (b) To find the components of F_J , write Newton's second law for both the x and y directions. Then combine them to find the magnitude.

$$\sum F_x = F_{Jx} - F_M \cos \theta = 0 \rightarrow F_{Jx} = F_M \cos \theta = (249.9 \text{ N}) \cos 15^\circ = 241.4 \text{ N}$$

$$\sum F_y = F_M \sin \theta - mg - F_{Jy} = 0 \rightarrow$$

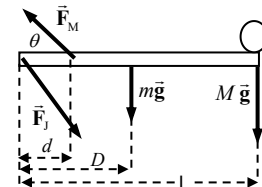
$$F_{Jy} = F_M \sin \theta - mg = (249.9 \text{ N}) \sin 15^\circ - (3.3 \text{ kg})(9.80 \text{ m/s}^2) = 32.3 \text{ N}$$

$$F_J = \sqrt{F_{Jx}^2 + F_{Jy}^2} = \sqrt{(241.4 \text{ N})^2 + (32.3 \text{ N})^2} = 243.6 \text{ N} \approx \boxed{240 \text{ N}}$$

33. Calculate the torques about the shoulder joint, which is at the left end of the free-body diagram of the arm. Since the arm is in equilibrium, the sum of the torques will be zero. Take counterclockwise torques to be positive. The force due to the shoulder joint is drawn, but it exerts no torque about the shoulder joint.

$$\sum \tau = F_M d \sin \theta - mgD - Mg \ell = 0$$

$$F_M = \frac{mD + M \ell}{d \sin \theta} g = \frac{(3.3 \text{ kg})(0.24 \text{ m}) + (8.5 \text{ kg})(0.52 \text{ m})}{(0.12 \text{ m}) \sin 15^\circ} (9.80 \text{ m/s}^2) = \boxed{1600 \text{ N}}$$

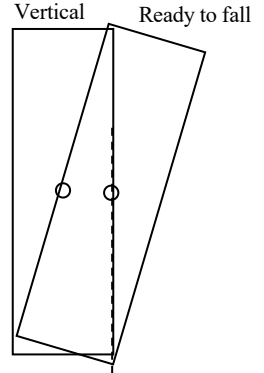


34. Take torques about the elbow joint. Let clockwise torques be positive. Since the arm is in equilibrium, the total torque will be 0.

$$\sum \tau = (2.0 \text{ kg})g(0.15 \text{ m}) + (25 \text{ kg})g(0.35 \text{ m}) - F_{\max}(0.050 \text{ m}) \sin 105^\circ = 0 \rightarrow$$

$$F_{\max} = \frac{(2.0 \text{ kg})g(0.15 \text{ m}) + (25 \text{ kg})g(0.35 \text{ m})}{(0.050 \text{ m}) \sin 105^\circ} = 1836 \text{ N} \approx \boxed{1800 \text{ N}}$$

35. From Section 9-4: “An object whose CG is above its base of support will be stable if a vertical line projected downward from the CG falls within the base of support.” For the tower, the base of support is a circle of radius 7.7 m. If the top is 4.5 m off center, then the CG will be 2.25 m off center, and a vertical line downward from the CG will be 2.25 m from the center of the base. As long as that vertical line is less than 7.7 m from the center of the base, the tower will be in stable equilibrium. To be unstable, the CG has to be more than 7.7 m off center, so the top must be more than $2 \times (7.7 \text{ m}) = 15.4 \text{ m}$ off center. Thus the top will have to lean $15.4 \text{ m} - 4.5 \text{ m} = \boxed{10.9 \text{ m}}$ farther to reach the verge of instability.



36. (a) The maximum distance for brick #1 to remain on brick #2 will be reached when the CM of brick #1 is directly over the edge of brick #2. Thus brick #1 will overhang brick #2 by $x_1 = \ell/2$.

The maximum distance for the top two bricks to remain on brick #3 will be reached when the center of mass of the top two bricks is directly over the edge of brick #3. The CM of the top two bricks is (obviously) at the point labeled X on brick #2, a distance of $\ell/4$ from the right edge of brick #2. Thus $x_2 = \ell/4$.

The maximum distance for the top three bricks to remain on brick #4 will be reached when the center of mass of the top three bricks is directly over the edge of brick #4. The CM of the top three bricks is at the point labeled X on brick #3 and is found relative to the

center of brick #3 by $\text{CM} = \frac{m(0) + 2m(\ell/2)}{3m} = \ell/3$, or $\ell/6$ from the right edge of brick #3. Thus $x_3 = \ell/6$.

The maximum distance for the four bricks to remain on a tabletop will be reached when the center of mass of the four bricks is directly over the edge of the table. The CM of all four bricks is at the point labeled X on brick #4 and is found relative to the center of brick #4 by

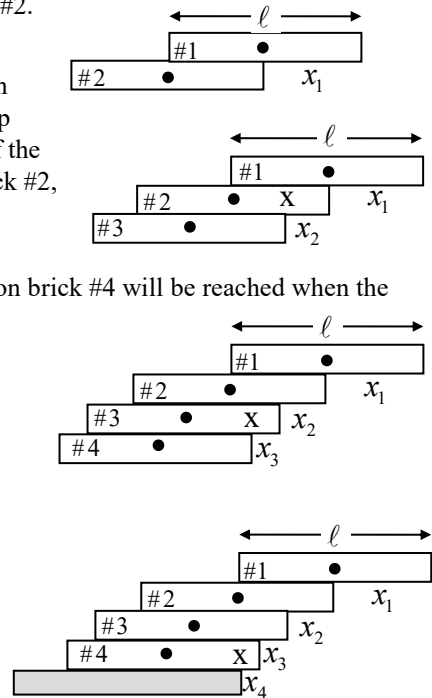
$$\text{CM} = \frac{m(0) + 3m(\ell/2)}{4m} = 3\ell/8, \text{ or } \ell/8 \text{ from the right edge}$$

of brick #4. Thus $x_4 = \ell/8$.

- (b) From the last diagram, the distance from the edge of the tabletop to the right edge of brick #1 is $x_4 + x_3 + x_2 + x_1 = (\ell/8) + (\ell/6) + (\ell/4) + (\ell/2) = 25\ell/24 > \ell$

Since this distance is greater than ℓ , the answer is yes, the first brick is completely beyond the edge of the table.

- (c) From the work in part (a), we see that the general formula for the total distance spanned by n bricks is



$$x_1 + x_2 + x_3 + \dots + x_n = (\ell/2) + (\ell/4) + (\ell/6) + \dots + (\ell/2n) = \sum_{i=1}^n \frac{\ell}{2i}$$

- (d) The arch is to span 1.0 m, so the span from one side will be 0.50 m. Thus, we must solve

$\sum_{i=1}^n \frac{0.30 \text{ m}}{2i} \geq 0.50 \text{ m}$. Evaluation of this expression for various values of n shows that 15 bricks will span a distance of 0.498 m and that 16 bricks will span a distance of 0.507 m. Thus, it takes 16 bricks for each half-span, plus 1 brick on top and 1 brick as the base on each side (as in Fig. 9-67b), for a total of **35 bricks**.

37. The amount of stretch can be found using the elastic modulus in Eq. 9-4.

$$\Delta \ell = \frac{1}{E} \frac{F}{A} \ell_0 = \frac{1}{3 \times 10^9 \text{ N/m}^2} \frac{275 \text{ N}}{\pi (5.00 \times 10^{-4})^2} (0.300 \text{ m}) = \boxed{3.50 \times 10^{-2} \text{ m}}$$

38. (a) $\text{stress} = \frac{F}{A} = \frac{mg}{A} = \frac{(25,000 \text{ kg})(9.80 \text{ m/s}^2)}{1.4 \text{ m}^2} = 175,000 \text{ N/m}^2 \approx \boxed{1.8 \times 10^5 \text{ N/m}^2}$

(b) $\text{strain} = \frac{\text{stress}}{\text{Young's modulus}} = \frac{175,000 \times 10^5 \text{ N/m}^2}{50 \times 10^9 \text{ N/m}^2} = \boxed{3.5 \times 10^{-6}}$

39. The change in length is found from the strain.

$$\text{strain} = \frac{\Delta \ell}{\ell_0} \rightarrow \Delta \ell = \ell_0 (\text{strain}) = (8.6 \text{ m})(3.5 \times 10^{-6}) = \boxed{3.0 \times 10^{-5} \text{ m}}$$

40. (a) $\text{stress} = \frac{F}{A} = \frac{mg}{A} = \frac{(1700 \text{ kg})(9.80 \text{ m/s}^2)}{0.012 \text{ m}^2} = 1.388 \times 10^6 \text{ N/m}^2 \approx \boxed{1.4 \times 10^6 \text{ N/m}^2}$

(b) $\text{strain} = \frac{\text{stress}}{\text{Young's modulus}} = \frac{1.388 \times 10^6 \text{ N/m}^2}{200 \times 10^9 \text{ N/m}^2} = 6.94 \times 10^{-6} \approx \boxed{6.9 \times 10^{-6}}$

(c) $\Delta \ell = (\text{strain})(\ell_0) = (6.94 \times 10^{-6})(9.50 \text{ m}) = 6.593 \times 10^{-5} \text{ m} \approx \boxed{6.6 \times 10^{-5} \text{ m}}$

41. The change in volume is given by Eq. 9-7. We assume the original pressure is atmospheric pressure, $1.0 \times 10^5 \text{ N/m}^2$.

$$\Delta V = -V_0 \frac{\Delta P}{B} = -(1000 \text{ cm}^3) \frac{(2.6 \times 10^6 \text{ N/m}^2 - 1.0 \times 10^5 \text{ N/m}^2)}{1.0 \times 10^9 \text{ N/m}^2} = -2.5 \text{ cm}^3$$

$$V = V_0 + \Delta V = 1000 \text{ cm}^3 - 2.5 \text{ cm}^3 = \boxed{997.5 \text{ cm}^3}$$

42. The relationship between pressure change and volume change is given by Eq. 9-7.

$$\Delta V = -V_0 \frac{\Delta P}{B} \rightarrow \Delta P = -\frac{\Delta V}{V_0} B = -(0.10 \times 10^{-2})(90 \times 10^9 \text{ N/m}^2) = \boxed{9.0 \times 10^7 \text{ N/m}^2}$$

$$\frac{\Delta P}{P_{\text{atm}}} = \frac{9.0 \times 10^7 \text{ N/m}^2}{1.0 \times 10^5 \text{ N/m}^2} = \boxed{9.0 \times 10^2}, \text{ or } 900 \text{ atmospheres}$$

43. The Young's modulus is the stress divided by the strain.

$$\text{Young's modulus} = \frac{\text{stress}}{\text{strain}} = \frac{F/A}{\Delta\ell/\ell_0} = \frac{(13.4 \text{ N})/\left[\pi\left(\frac{1}{2}\times 8.5\times 10^{-3} \text{ m}\right)^2\right]}{(3.7\times 10^{-3} \text{ m})/(15\times 10^{-2} \text{ m})} = \boxed{9.6\times 10^6 \text{ N/m}^2}$$

44. The mass can be calculated from the equation for the relationship between stress and strain. The force causing the strain is the weight of the mass suspended from the wire. Use Eq. 9-4.

$$\frac{\Delta\ell}{\ell_0} = \frac{1}{E} \frac{F}{A} = \frac{mg}{EA} \rightarrow m = \frac{EA \Delta\ell}{g \ell_0} = (200\times 10^9 \text{ N/m}^2) \frac{\pi(1.15\times 10^{-3} \text{ m})^2 0.030}{(9.80 \text{ m/s}^2)} = \boxed{25 \text{ kg}}$$

45. The percentage change in volume is found by multiplying the relative change in volume by 100. The change in pressure is 199 times atmospheric pressure, since it increases from atmospheric pressure to 200 times atmospheric pressure. Use Eq. 9-7.

$$100 \frac{\Delta V}{V_0} = -100 \frac{\Delta P}{B} = -100 \frac{199(1.0\times 10^5 \text{ N/m}^2)}{90\times 10^9 \text{ N/m}^2} = \boxed{-2\times 10^{-2},}$$

The negative sign indicates that the interior space got smaller.

46. Set the compressive strength of the bone equal to the stress of the bone.

$$\text{compressive strength} = \frac{F_{\max}}{A} \rightarrow F_{\max} = (170\times 10^6 \text{ N/m}^2)(3.0\times 10^{-4} \text{ m}^2) = \boxed{5.1\times 10^4 \text{ N}}$$

47. (a) The maximum tension can be found from the ultimate tensile strength of the material.

$$\text{tensile strength} = \frac{F_{\max}}{A} \rightarrow$$

$$F_{\max} = (\text{tensile strength})A = (500\times 10^6 \text{ N/m}^2)\pi(5.00\times 10^{-4} \text{ m})^2 = \boxed{393 \text{ N}}$$

- (b) To prevent breakage, **thicker strings** should be used, which will increase the cross-sectional area of the strings and thus increase the maximum force. Breakage occurs because when the strings are hit by the ball, they stretch, increasing the tension. The strings are reasonably tight in the normal racket configuration, so when the tension is increased by a particularly hard hit, the tension may exceed the maximum force.

48. (a) The area can be found from the ultimate tensile strength of the material.

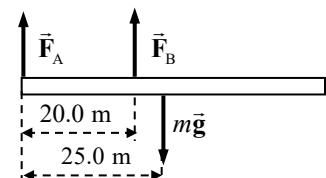
$$\frac{\text{tensile strength}}{\text{safety factor}} = \frac{F}{A} \rightarrow A = F \left(\frac{\text{safety factor}}{\text{tensile strength}} \right) \rightarrow$$

$$A = (270 \text{ kg})(9.80 \text{ m/s}^2) \frac{7.0}{500\times 10^6 \text{ N/m}^2} = 3.704\times 10^{-5} \text{ m}^2 \approx \boxed{3.7\times 10^{-5} \text{ m}^2}$$

- (b) The change in length can be found from the stress-strain relationship, Eq. 9-4.

$$\frac{F}{A} = E \frac{\Delta\ell}{\ell_0} \rightarrow \Delta\ell = \frac{\ell_0 F}{AE} = \frac{(7.5 \text{ m})(320 \text{ kg})(9.80 \text{ m/s}^2)}{(3.704\times 10^{-5} \text{ m}^2)(200\times 10^9 \text{ N/m}^2)} = \boxed{2.7\times 10^{-3} \text{ m}}$$

49. For each support, to find the minimum cross-sectional area with a safety factor means that $\frac{F}{A} = \frac{\text{strength}}{\text{safety factor}}$, where either the tensile or compressive strength is used, as appropriate for each force. To find the force on each support, use the conditions of equilibrium for the beam.



Take torques about the left end of the beam, calling counterclockwise torques positive, and also sum the vertical forces, taking upward forces as positive.

$$\sum \tau = F_B(20.0 \text{ m}) - mg(25.0 \text{ m}) = 0 \rightarrow F_B = \frac{25.0}{20.0}mg = 1.25mg$$

$$\sum F_y = F_A + F_B - mg = 0 \rightarrow F_A = mg - F_B = mg - 1.25mg = -0.25mg$$

Notice that the forces on the supports are the opposite of \vec{F}_A and \vec{F}_B . So the force on support A is directed upward, which means that support A is in tension. The force on support B is directed downward, so support B is in compression.

$$\frac{F_A}{A_A} = \frac{\text{tensile strength}}{9.0} \rightarrow$$

$$A_A = 9.0 \frac{(0.25mg)}{\text{tensile strength}} = 9.0 \frac{(0.25)(2.9 \times 10^3 \text{ kg})(9.80 \text{ m/s}^2)}{40 \times 10^6 \text{ N/m}^2} = \boxed{1.6 \times 10^{-3} \text{ m}^2}$$

$$\frac{F_B}{A_B} = \frac{\text{compressive strength}}{9.0} \rightarrow$$

$$A_B = 9.0 \frac{(1.25mg)}{\text{compressive strength}} = 9.0 \frac{(1.25)(2.9 \times 10^3 \text{ kg})(9.80 \text{ m/s}^2)}{35 \times 10^6 \text{ N/m}^2} = \boxed{9.1 \times 10^{-3} \text{ m}^2}$$

50. The maximum shear stress is to be 1/7th of the shear strength for iron. The maximum stress will occur for the minimum area and thus the minimum diameter.

$$\text{stress}_{\max} = \frac{F}{A_{\min}} = \frac{\text{shear strength}}{7.0} \rightarrow A = \pi \left(\frac{1}{2}d \right)^2 = \frac{7.0 F}{\text{shear strength}} \rightarrow$$

$$d = \sqrt{\frac{4(7.0)F}{\pi(\text{shear strength})}} = \sqrt{\frac{28(3300 \text{ N})}{\pi(170 \times 10^6 \text{ N/m}^2)}} = 1.3 \times 10^{-2} \text{ m} = \boxed{1.3 \text{ cm}}$$

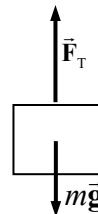
51. From the free-body diagram, write Newton's second law for the vertical direction. Solve for the maximum tension required in the cable, which will occur for an upward acceleration.

$$\sum F_y = F_T - mg = ma \rightarrow F_T = m(g + a)$$

The maximum stress is to be 1/8th of the tensile strength for steel. The maximum stress will occur for the minimum area and thus the minimum diameter.

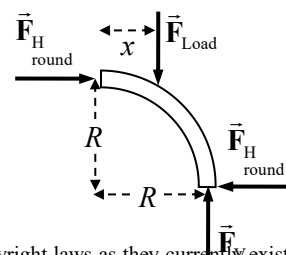
$$\text{stress}_{\max} = \frac{F_T}{A_{\min}} = \frac{\text{tensile strength}}{8.0} \rightarrow A = \pi \left(\frac{1}{2}d \right)^2 = \frac{8.0 F_T}{\text{tensile strength}} \rightarrow$$

$$d = \sqrt{\frac{4(8.0)m(g + a)}{\pi(\text{tensile strength})}} = \sqrt{\frac{32(3100 \text{ kg})(11.6 \text{ m/s}^2)}{\pi(500 \times 10^6 \text{ N/m}^2)}} = 2.71 \times 10^{-2} \text{ m} \approx \boxed{2.7 \text{ cm}}$$



52. Draw free-body diagrams similar to Figs. 9–31a and 9–31b for the forces on the right half of a round arch and a pointed arch. The load force is placed at the same horizontal position on each arch. For each half-arch, take torques about the lower right-hand corner, with counterclockwise as positive.

For the round arch:



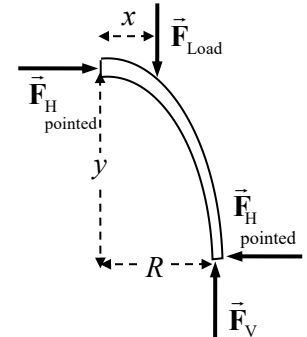
$$\sum \tau = F_{\text{Load}}(R-x) - F_{\text{H round}} R = 0 \rightarrow F_{\text{H round}} = F_{\text{Load}} \frac{R-x}{R}$$

For the pointed arch:

$$\sum \tau = F_{\text{Load}}(R-x) - F_{\text{H pointed}} y = 0 \rightarrow F_{\text{H pointed}} = F_{\text{Load}} \frac{R-x}{y}$$

Solve for y , given that $F_{\text{H pointed}} = \frac{1}{3} F_{\text{H round}}$.

$$F_{\text{H pointed}} = \frac{1}{3} F_{\text{H round}} \rightarrow F_{\text{Load}} \frac{R-x}{y} = \frac{1}{3} F_{\text{Load}} \frac{R-x}{R} \rightarrow y = 3R = 3\left(\frac{1}{2} 8.0 \text{ m}\right) = \boxed{12 \text{ m}}$$

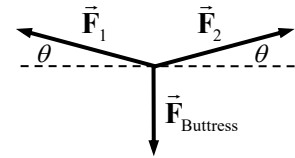


53. Write Newton's second law for the horizontal direction.

$$\sum F_x = F_2 \cos \theta - F_1 \cos \theta = 0 \rightarrow F_2 = F_1$$

Thus the two forces are the same size. Now write Newton's second law for the vertical direction.

$$\sum F_y = F_1 \sin \theta + F_1 \sin \theta - F_{\text{Buttress}} = 0 \rightarrow F_1 = \frac{F_{\text{Buttress}}}{2 \sin \theta} = \frac{4.2 \times 10^5 \text{ N}}{2 (\sin 5^\circ)} = \boxed{2.4 \times 10^6 \text{ N}}$$



54. (a) The pole will exert a downward force and a clockwise torque about the woman's right hand. Thus there must be an upward force exerted by the left hand to cause a counterclockwise torque for the pole to have a net torque of zero about the right hand. The force exerted by the right hand is then of such a magnitude and direction for the net vertical force on the pole to be zero.

$$\sum \tau = F_{\text{Left}}(0.32 \text{ m}) - mg(1.0 \text{ m}) = 0 \rightarrow$$

$$F_{\text{Left}} = mg \left(\frac{1.0 \text{ m}}{0.32 \text{ m}} \right) = \frac{(10.0 \text{ kg})(9.80 \text{ m/s}^2)}{0.32} = 306.25 \text{ N} \approx \boxed{310 \text{ N, upward}}$$

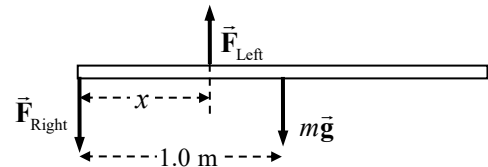
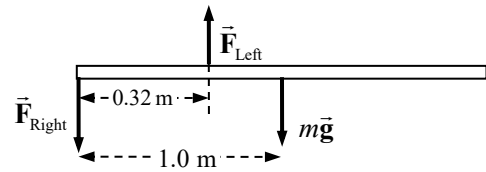
$$\sum F_y = F_{\text{Left}} - F_{\text{Right}} - mg = 0 \rightarrow$$

$$F_{\text{Right}} = F_{\text{Left}} - mg = 306.25 \text{ N} - (10.0 \text{ kg})(9.80 \text{ m/s}^2) = 208.25 \text{ N} \approx \boxed{210 \text{ N, downward}}$$

- (b) We see that the force due to the left hand is larger than the force due to the right hand, since both the right hand and gravity are downward. Set the left hand force equal to 150 N and calculate the location of the left hand by setting the net torque equal to zero.

$$\sum \tau = F_{\text{Left}} x - mg(1.0 \text{ m}) = 0 \rightarrow x = \frac{mg}{F_{\text{Left}}}(1.0 \text{ m}) = \frac{98.0 \text{ N}}{150 \text{ N}}(1.0 \text{ m}) = \boxed{0.65 \text{ m}}$$

As a check, calculate the force due to the right hand.



$$F_{\text{Right}} = F_{\text{Left}} - mg = 150 \text{ N} - 98.0 \text{ N} = 52 \text{ N} \quad \text{OK}$$

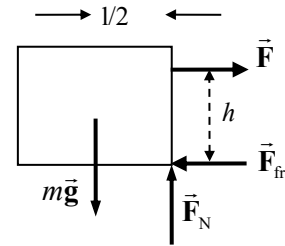
(c) Follow the same procedure, setting the left-hand force equal to 85 N:

$$\sum \tau = F_{\text{Left}}x - mg(1.0 \text{ m}) = 0 \rightarrow x = \frac{mg}{F_{\text{Left}}}(1.0 \text{ m}) = \frac{98.0 \text{ N}}{85 \text{ N}}(1.0 \text{ m}) = 1.153 \text{ m} \approx \boxed{1.2 \text{ m}}$$

$$F_{\text{Right}} = F_{\text{Left}} - mg = 85 \text{ N} - 98.0 \text{ N} = -13 \text{ N} \quad \text{OK}$$

Note that now the force due to the right hand must be pulling upward, because the left hand is on the opposite side of the center of the pole.

55. If the block is on the verge of tipping, the normal force will be acting at the lower right-hand corner of the block, as shown in the free-body diagram. The block will begin to rotate when the torque caused by the pulling force is larger than the torque caused by gravity. For the block to be able to slide the pulling force must be as large as the maximum static frictional force. Write the equations of equilibrium for forces in the x and y directions and for torque with the conditions as stated above.



$$\sum F_y = F_N - mg = 0 \rightarrow F_N = mg$$

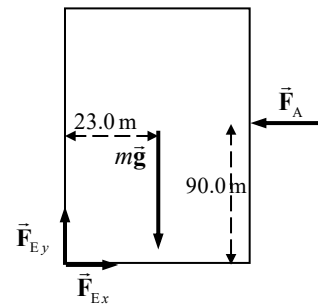
$$\sum F_x = F - F_{fr} = 0 \rightarrow F = F_{fr} = \mu_s F_N = \mu_s mg$$

$$\sum \tau = mg \frac{\ell}{2} - Fh = 0 \rightarrow \frac{mg\ell}{2} = Fh = \mu_s mgh$$

Solve for the coefficient of friction in this limiting case, to find $\mu_s = \frac{\ell}{2h}$.

- (a) If $\mu_s < \ell/2h$, then sliding will happen before tipping.
 (b) If $\mu_s > \ell/2h$, then tipping will happen before sliding.

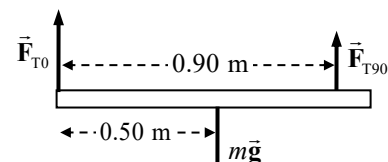
56. Assume that the building has just begun to tip, so that it is essentially vertical, but that all of the force on the building due to contact with the Earth is at the lower left-hand corner, as shown in the figure. Take torques about that corner, with counterclockwise torques as positive.



$$\begin{aligned} \sum \tau &= F_A(90.0 \text{ m}) - mg(23.0 \text{ m}) \\ &= [(950 \text{ N/m}^2)(180.0 \text{ m})(76.0 \text{ m})](90.0 \text{ m}) \\ &\quad - (1.8 \times 10^7 \text{ kg})(9.80 \text{ m/s}^2)(23.0 \text{ m}) = \boxed{-2.9 \times 10^9 \text{ m} \cdot \text{N}} \end{aligned}$$

Since this is a negative torque, the building will tend to rotate clockwise, which means it will rotate back down to the ground. Thus the building will not topple.

57. (a) The meter stick is in equilibrium, so both the net torque and the net force are zero. From the force diagram, write an expression for the net torque about the 90-cm mark, with counterclockwise torques as positive.



$$\sum \tau = mg(0.40 \text{ m}) - F_{T0}(0.90 \text{ m}) = 0 \rightarrow$$

$$F_{T0} = mg \frac{0.40}{0.90} = (0.180 \text{ kg})(9.80 \text{ m/s}^2) \frac{0.40}{0.90} = \boxed{0.78 \text{ N}}$$

- (b) Write Newton's second law for the vertical direction with a net force of 0 to find the other tension.

$$\sum F_y = F_{T0} + F_{T90} - mg = 0 \rightarrow$$

$$F_{T90} = mg - F_{T0} = (0.180 \text{ kg})(9.80 \text{ m/s}^2) - 0.78 \text{ N} = \boxed{0.98 \text{ N}}$$

58. The maximum compressive force in a column will occur at the bottom. The bottom layer supports the entire weight of the column, so the compressive force on that layer is mg . For the column to be on the verge of buckling, the weight divided by the area of the column will be the compressive strength of the material. The mass of the column is its volume (area \times height) times its density.

$$\frac{mg}{A} = \text{compressive strength} = \frac{hA\rho g}{A} \rightarrow h = \frac{\text{compressive strength}}{\rho g}$$

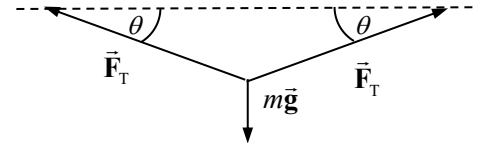
Note that the area of the column cancels out of the expression, so the height does not depend on the cross-sectional area of the column.

$$(a) \quad h_{\text{steel}} = \frac{\text{compressive strength}}{\rho g} = \frac{500 \times 10^6 \text{ N/m}^2}{(7.8 \times 10^3 \text{ kg/m}^3)(9.80 \text{ m/s}^2)} = \boxed{6500 \text{ m}}$$

$$(b) \quad h_{\text{granite}} = \frac{\text{compressive strength}}{\rho g} = \frac{170 \times 10^6 \text{ N/m}^2}{(2.7 \times 10^3 \text{ kg/m}^3)(9.80 \text{ m/s}^2)} = \boxed{6400 \text{ m}}$$

59. The radius of the wire can be determined from the relationship between stress and strain, expressed by Eq. 9-5.

$$\frac{F}{A} = E \frac{\Delta \ell}{\ell_0} \rightarrow A = \frac{F \ell_0}{E \Delta \ell} = \pi r^2 \rightarrow r = \sqrt{\frac{1}{\pi} \frac{F \ell_0}{E \Delta \ell}}$$

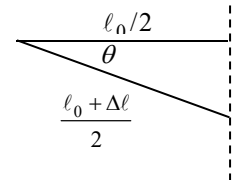


Use the free-body diagram for the attachment point of the mass and wire to get the wire's tension.

$$\sum F_y = 2F_T \sin \theta - mg = 0 \rightarrow F_T = \frac{mg}{2 \sin \theta} = \frac{(25 \text{ kg})(9.80 \text{ m/s}^2)}{2 \sin 12^\circ} = 589.2 \text{ N}$$

The fractional change in the length of the wire can be found from the geometry of the problem, as seen in the second diagram.

$$\cos \theta = \frac{\ell_0/2}{\frac{\ell_0 + \Delta \ell}{2}} \rightarrow \frac{\Delta \ell}{\ell_0} = \frac{1}{\cos \theta} - 1 = \frac{1}{\cos 12^\circ} - 1 = 2.234 \times 10^{-2}$$



Thus, the radius is

$$r = \sqrt{\frac{1}{\pi} \frac{F_T \ell_0}{E \Delta \ell}} = \sqrt{\frac{1}{\pi} \left(\frac{589.2 \text{ N}}{70 \times 10^9 \text{ N/m}^2} \right) \frac{1}{(2.234 \times 10^{-2})}} = \boxed{3.5 \times 10^{-4} \text{ m}}$$

60. The limiting condition for the painter's safety is the tension in the ropes. The ropes can exert only an upward tension on the scaffold. The tension will be least in the rope that is farther from the painter. The mass of the pail is m_p , the mass of the scaffold is m , and the mass of the painter is M .

Find the distance to the right that the painter can walk before the tension in the left rope becomes zero. Take torques about the point where the right-side rope is attached to the scaffold, so that its value need not be known. Take counterclockwise torques as positive.

$$\sum \tau = mg(2.0 \text{ m}) + m_p g(3.0 \text{ m}) - Mgx = 0 \rightarrow$$

$$x = \frac{m(2.0 \text{ m}) + m_p(3.0 \text{ m})}{M} = \frac{(25 \text{ kg})(2.0 \text{ m}) + (4.0 \text{ kg})(3.0 \text{ m})}{65.0 \text{ kg}} = 0.9538 \text{ m} \approx 0.95 \text{ m}$$

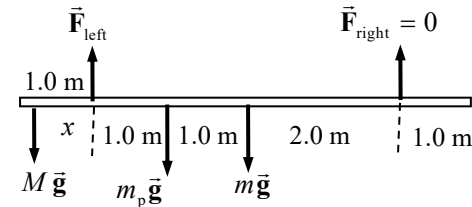
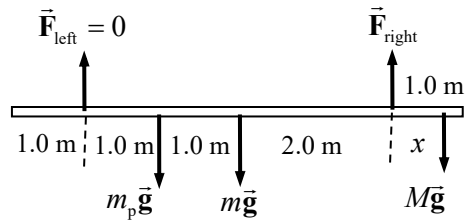
The painter can walk to within 5 cm of the right edge of the scaffold.

Now find the distance to the left that the painter can walk before the tension in the right rope becomes zero. Take torques about the point where the left-side tension is attached to the scaffold, so that its value need not be known. Take counterclockwise torques as positive.

$$\sum \tau = Mgx - m_p g(1.0 \text{ m}) - mg(2.0 \text{ m}) = 0 \rightarrow$$

$$x = \frac{m(2.0 \text{ m}) + m_p(1.0 \text{ m})}{M} = \frac{(25 \text{ kg})(2.0 \text{ m}) + (4.0 \text{ kg})(1.0 \text{ m})}{65.0 \text{ kg}} = 0.8308 \text{ m} \approx 0.83 \text{ m}$$

The painter can walk to within 17 cm of the left edge of the scaffold. Both ends are dangerous.



61. See the free-body diagram. The ball is at rest, so it is in equilibrium. Write Newton's second law for the horizontal and vertical directions, and solve for the forces.

$$\sum F_{\text{horiz}} = F_B \sin \theta_B - F_A \sin \theta_A = 0 \rightarrow F_B = F_A \frac{\sin \theta_A}{\sin \theta_B}$$

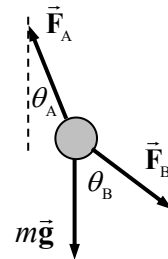
$$\sum F_{\text{vert}} = F_A \cos \theta_A - F_B \cos \theta_B - mg = 0 \rightarrow F_A \cos \theta_A = F_B \cos \theta_B + mg \rightarrow$$

$$F_A \cos \theta_A = F_A \frac{\sin \theta_A}{\sin \theta_B} \cos \theta_B + mg \rightarrow F_A \left(\cos \theta_A - \frac{\sin \theta_A \cos \theta_B}{\sin \theta_B} \right) = mg \rightarrow$$

$$F_A = mg \frac{\sin \theta_B}{(\cos \theta_A \sin \theta_B - \sin \theta_A \cos \theta_B)} = mg \frac{\sin \theta_B}{\sin (\theta_B - \theta_A)} = (15.0 \text{ kg})(9.80 \text{ m/s}^2) \frac{\sin 53^\circ}{\sin 31^\circ}$$

$$= 228 \text{ N} \approx \boxed{230 \text{ N}}$$

$$F_B = F_A \frac{\sin \theta_A}{\sin \theta_B} = (228 \text{ N}) \frac{\sin 22^\circ}{\sin 53^\circ} = 107 \text{ N} \approx \boxed{110 \text{ N}}$$



62. The number of supports can be found from the compressive strength of the wood. Since the wood will be oriented longitudinally, the stress will be parallel to the grain.

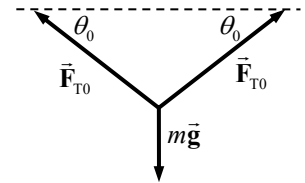
$$\begin{aligned} \frac{\text{compressive strength}}{\text{safety factor}} &= \frac{\text{load force on supports}}{\text{area of supports}} = \frac{\text{weight of roof}}{(\text{number of supports})(\text{area per support})} \\ (\text{number of supports}) &= \frac{\text{weight of roof}}{(\text{area per support})} \frac{\text{safety factor}}{\text{compressive strength}} \\ &= \frac{(1.36 \times 10^4 \text{ kg})(9.80 \text{ m/s}^2)}{(0.040 \text{ m})(0.090 \text{ m})} \frac{12}{(35 \times 10^6 \text{ N/m}^2)} = 12.69 \text{ supports} \end{aligned}$$

Since there are to be more than 12 supports, and to have the same number of supports on each side, there will be 14 supports, or 7 supports on each side. That means there will be 6 support-to-support

$$\text{spans, each of which would be given by } \text{spacing} = \frac{10.0 \text{ m}}{6 \text{ gaps}} = \boxed{1.66 \text{ m/gap}}$$

63. Since the backpack is midway between the two trees, the angles in the free-body diagram are equal. Write Newton's second law for the vertical direction for the point at which the backpack is attached to the cord, with the weight of the backpack being the original downward vertical force.

$$\sum F_y = 2F_{T0} \sin \theta_0 - mg = 0 \rightarrow F_{T0} = \frac{mg}{2 \sin \theta_0}$$

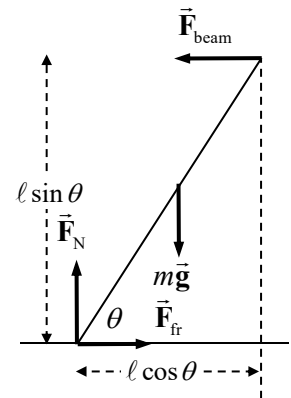


Now assume the bear pulls down with an additional force, F_{bear} . The force equation would be modified as follows:

$$\begin{aligned} \sum F_y &= 2F_{T \text{ final}} \sin \theta_{\text{final}} - mg - F_{\text{bear}} = 0 \rightarrow \\ F_{\text{bear}} &= 2F_{T \text{ final}} \sin \theta_{\text{final}} - mg = 2(2F_{T0}) \sin \theta_{\text{final}} - mg = 4 \left(\frac{mg}{2 \sin \theta_0} \right) \sin \theta_{\text{final}} - mg \\ &= mg \left(\frac{2 \sin \theta_{\text{final}}}{\sin \theta_0} - 1 \right) = (23.0 \text{ kg})(9.80 \text{ m/s}^2) \left(\frac{2 \sin 27^\circ}{\sin 15^\circ} - 1 \right) = 565.3 \text{ N} \approx \boxed{570 \text{ N}} \end{aligned}$$

64. Draw a free-body diagram for one of the beams. By Newton's third law, if the right beam pushes down on the left beam, then the left beam pushes up on the right beam. But the geometry is symmetric for the two beams, so the beam contact force must be horizontal. For the beam to be in equilibrium, $F_N = mg$, and $F_{\text{fr}} = \mu_s F_N = \mu mg$ is the maximum friction force. Take torques about the top of the beam, so that \vec{F}_{beam} exerts no torque. Let clockwise torques be positive.

$$\begin{aligned} \sum \tau &= F_N \ell \cos \theta - mg \left(\frac{1}{2} \ell \right) \cos \theta - F_{\text{fr}} \ell \sin \theta = 0 \rightarrow \\ \theta &= \tan^{-1} \frac{1}{2\mu_s} = \tan^{-1} \frac{1}{2(0.5)} = \boxed{45^\circ} \end{aligned}$$



65. (a) The fractional decrease in the rod's length is the strain. Use Eq. 9-4. The force applied is the weight of the man.

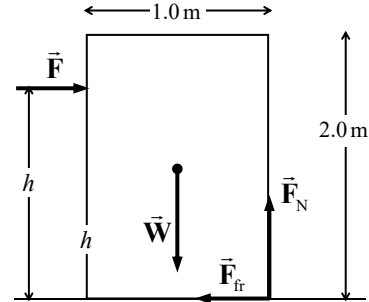
$$\frac{\Delta \ell}{\ell_0} = \frac{F}{AE} = \frac{mg}{\pi r^2 E} = \frac{(65 \text{ kg})(9.80 \text{ m/s}^2)}{\pi (0.15)^2 (200 \times 10^9 \text{ N/m}^2)} = 4.506 \times 10^{-8} = \boxed{(4.5 \times 10^{-6})\%}$$

- (b) The fractional change is the same for the atoms as for the macroscopic material. Let d represent the interatomic spacing.

$$\frac{\Delta d}{d_0} = \frac{\Delta \ell}{\ell_0} = 4.506 \times 10^{-8} \rightarrow$$

$$\Delta d = (4.506 \times 10^{-8})d_0 = (4.506 \times 10^{-8})(2.0 \times 10^{-10} \text{ m}) = \boxed{9.0 \times 10^{-18} \text{ m}}$$

66. Consider the free-body diagram for the box. The box is assumed to be in equilibrium, but just on the verge of both sliding and tipping. Since it is on the verge of sliding, the static frictional force is at its maximum value. Use the equations of equilibrium. Take torques about the lower right-hand corner where the box touches the floor, and take clockwise torques as positive. We assume that the box is just barely tipped up on its corner, so that the forces are still parallel and perpendicular to the edges of the box.



$$\sum F_y = F_N - W = 0 \rightarrow F_N = W$$

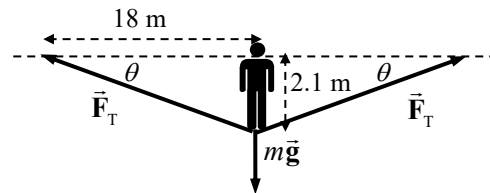
$$\sum F_x = F - F_{fr} = 0 \rightarrow F = F_{fr} = \mu W = (0.60)(250 \text{ N}) = \boxed{150 \text{ N}}$$

$$\sum \tau = Fh - W(0.5 \text{ m}) = 0 \rightarrow h = (0.5 \text{ m}) \frac{W}{F} = (0.5 \text{ m}) \frac{250 \text{ N}}{150 \text{ N}} = \boxed{0.83 \text{ m}}$$

67. From the free-body diagram (not to scale), write the force equilibrium condition for the vertical direction.

$$\sum F_y = 2F_T \sin \theta - mg = 0$$

$$F_T = \frac{mg}{2 \sin \theta} \approx \frac{mg}{2 \tan \theta} = \frac{(60.0 \text{ kg})(9.80 \text{ m/s}^2)}{2 \left(\frac{2.1 \text{ m}}{18 \text{ m}} \right)} = \boxed{2500 \text{ N}}$$



Note that the angle is small enough (about 7°) that we have made the substitution $\sin \theta \approx \tan \theta$.

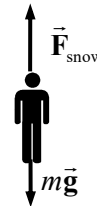
It is not possible to increase the tension so that there is no sag. There must always be a vertical component of the tension to balance the gravity force. The larger the tension gets, the smaller the sag angle will be, however.

68. Assume a constant acceleration as the person is brought to rest, with up as the positive direction. Use Eq. 2-11c to find the acceleration. From the acceleration, find the average force of the snow on the person, and compare the force per area with the strength of body tissue. From the free-body diagram, we have $F_{\text{snow}} - mg = ma \rightarrow F_{\text{snow}} = m(a + g)$.

$$v^2 = v_0^2 - 2a(x - x_0) \rightarrow a = \frac{v^2 - v_0^2}{2(x - x_0)} = \frac{0 - (55 \text{ m/s})^2}{2(-1.0 \text{ m})} = 1513 \text{ m/s}^2$$

$$\frac{F_{\text{snow}}}{A} = \frac{m(a + g)}{A} = \frac{(75 \text{ kg})(1513 \text{ m/s}^2 + 9.80 \text{ m/s}^2)}{0.30 \text{ m}^2} = 3.81 \times 10^5 \text{ N/m}^2$$

$$\frac{F_{\text{snow}}}{A} < \text{tissue strength} = 5 \times 10^5 \text{ N/m}^2$$



Since the average force on the person is less than the strength of body tissue, the person may escape serious injury. Certain parts of the body, such as the legs if landing feet first, may get more than the average force, though, and still sustain injury.

69. The force in the left vertical support column is 44,100 N, in compression. We want a steel column that can handle three times that, or 132,300 N. Steel has a compressive strength of $500 \times 10^6 \text{ N/m}^2$. Use this to find the area.

$$\frac{F}{A} = \frac{132,300 \text{ N}}{A} = 500 \times 10^6 \text{ N/m}^2 \rightarrow A = \frac{132,300 \text{ N}}{500 \times 10^6 \text{ N/m}^2} = 2.646 \times 10^{-4} \text{ m}^2 \approx \boxed{2.6 \times 10^{-4} \text{ m}^2}$$

If the column were square, each side would be 1.6 cm. If the column were cylindrical, the radius would be 9.2 mm.

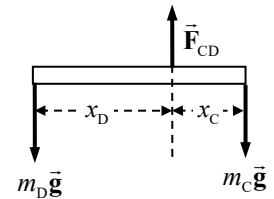
70. Each crossbar in the mobile is in equilibrium, so the net torque about the suspension point for each crossbar must be 0. Counterclockwise torques will be taken as positive. The suspension point is used so that the tension in the suspension string need not be known initially. The net vertical force must also be 0.

The bottom bar:

$$\sum \tau = m_D g x_D - m_C g x_C = 0 \rightarrow$$

$$m_C = m_D \frac{x_D}{x_C} = m_D \frac{17.50 \text{ cm}}{5.00 \text{ cm}} = 3.50 m_D$$

$$\sum F_y = F_{CD} - m_C g - m_D g = 0 \rightarrow F_{CD} = (m_C + m_D)g = 4.50 m_D g$$



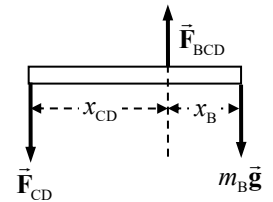
The middle bar:

$$\sum \tau = F_{CD} x_{CD} - m_B g x_B = 0 \rightarrow F_{CD} = m_B g \frac{x_B}{x_{CD}} \rightarrow 4.50 m_D g = m_B g \frac{x_B}{x_{CD}}$$

$$m_D = \frac{m_B}{4.50} \frac{x_B}{x_{CD}} = \frac{(0.748 \text{ kg})(5.00 \text{ cm})}{(4.50)(15.00 \text{ cm})} = 0.05541 \approx \boxed{5.54 \times 10^{-2} \text{ kg}}$$

$$m_C = 3.50 m_D = (3.50)(0.05541 \text{ kg}) = \boxed{0.194 \text{ kg}}$$

$$\sum F_y = F_{BCD} - F_{CD} - m_B g = 0 \rightarrow F_{BCD} = F_{CD} + m_B g = (4.50 m_D + m_B)g$$

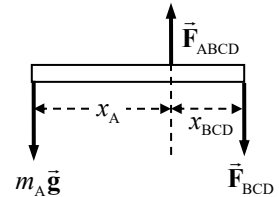


The top bar:

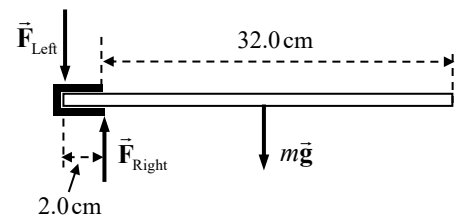
$$\sum \tau = m_A g x_A - F_{BCD} x_{BCD} = 0 \rightarrow$$

$$m_A = \frac{(4.50 m_D + m_B)g x_{BCD}}{g x_A} = (4.50 m_D + m_B) \frac{x_{BCD}}{x_A}$$

$$= [(4.50)(0.05541 \text{ kg}) + 0.748 \text{ kg}] \frac{7.50 \text{ cm}}{30.00 \text{ cm}} = \boxed{0.249 \text{ kg}}$$



71. (a) The weight of the shelf exerts a downward force and a clockwise torque about the point where the shelf touches the wall. Thus, there must be an upward force and a counterclockwise torque exerted by the slot for the shelf to be in equilibrium. Since any force exerted by the slot will have a short lever arm relative to the point where the shelf touches the wall, the upward force must be larger than the gravity force. Accordingly, there then must be a downward force exerted by the slot at its left edge, exerting no torque, but balancing the vertical forces.
- (b) Calculate the values of the three forces by first taking torques about the left end of the shelf, with the net torque being zero, and then sum the vertical forces, with the sum being zero.



$$\sum \tau = F_{\text{right}}(0.020 \text{ m}) - mg(0.170 \text{ m}) = 0 \rightarrow$$

$$F_{\text{right}} = (6.6 \text{ kg})(9.80 \text{ m/s}^2) \left(\frac{0.170 \text{ m}}{0.020 \text{ m}} \right) = 549.8 \text{ N} \approx \boxed{550 \text{ N}}$$

$$\sum F_y = F_{\text{right}} - F_{\text{left}} - mg \rightarrow$$

$$F_{\text{left}} = F_{\text{right}} - mg = 549.8 \text{ N} - (6.6 \text{ kg})(9.80 \text{ m/s}^2) = \boxed{490 \text{ N}}$$

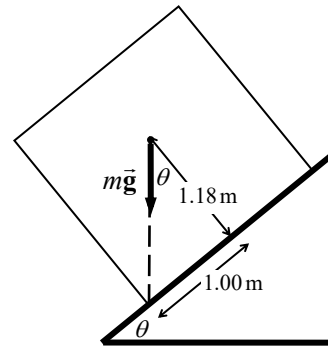
$$mg = (6.6 \text{ kg})(9.80 \text{ m/s}^2) = \boxed{65 \text{ N}}$$

(c) The torque exerted by the support about the left end of the rod is

$$\tau = F_{\text{right}}(2.0 \times 10^{-2} \text{ m}) = (549.8 \text{ N})(2.0 \times 10^{-2} \text{ m}) = \boxed{11 \text{ m} \cdot \text{N}}$$

72. See the free-body diagram for the crate on the verge of tipping. From Fig. 9-16 and the associated discussion, if a vertical line projected downward from the center of gravity falls outside the base of support, then the object will topple. So the limiting case is for the vertical line to intersect the edge of the base of support. Any more tilting and the gravity force would cause the block to tip over, with the axis of rotation through the lower corner of the crate.

$$\tan \theta = \frac{1.00}{1.18} \rightarrow \theta = \tan^{-1} \frac{1.00}{1.18} = \boxed{40^\circ} \text{ (2 significant figures)}$$



The other forces on the block, the normal force and the frictional force, would act at the lower corner. They would cause no torque about the lower corner. The gravity force causes the tipping.

Solutions to Search and Learn Problems

1. (a) Use conservation of energy to determine the speed when the person reaches the ground. Set the potential energy of the ground as zero ($y = 0$).

$$KE_1 + PE_1 = KE_2 + PE_2 \rightarrow 0 + mgy_1 = \frac{1}{2}mv^2 + 0$$

$$v = \sqrt{2gy_1} = \sqrt{2(9.8 \text{ m/s}^2)(3.0 \text{ m})} = 7.668 \text{ m/s} \approx \boxed{7.7 \text{ m/s}}$$

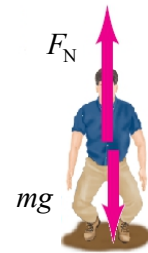
- (b) When the person reaches the ground, two forces will act on him: the force of gravity pulling down and the normal force of the ground pushing up. The sum of these two forces provides the net decelerating force. The net work done during deceleration is equal to the change in kinetic energy.

$$\sum Fd = \Delta KE \rightarrow (mg - F_N)(d) = 0 - \frac{1}{2}mv^2$$

$$F_N = mg + \frac{mv^2}{2d} = (65 \text{ kg})(9.8 \text{ m/s}^2) + \frac{(65 \text{ kg})(7.668 \text{ m/s})^2}{2(0.50 \text{ m})} = 4459 \text{ N} \approx \boxed{4500 \text{ N}}$$

- (c) Repeat the previous calculation for a stopping distance of $d = 0.010 \text{ m}$.

$$F_N = mg + \frac{mv^2}{2d} = (65 \text{ kg})(9.8 \text{ m/s}^2) + \frac{(65 \text{ kg})(7.668 \text{ m/s})^2}{2(0.010 \text{ m})} = 1.917 \times 10^5 \text{ N} \approx \boxed{1.9 \times 10^5 \text{ N}}$$



- (d) The force is evenly spread between each leg, so divide half of the force by the area of the tibia to determine the stress. Then compare this stress to the compressive strength of the tibia given in Table 9-2.

$$\frac{F}{A} = \frac{\frac{1}{2}(4459 \text{ N})}{3.0 \times 10^{-4} \text{ m}^2} = \boxed{7.4 \times 10^6 \text{ N/m}^2} < 170 \times 10^6 \text{ N/m}^2$$

The stress is much less than the compressive strength, so it is unlikely that the tibia will break.

- (e) Repeating the calculation for the distance of 0.010 m:

$$\frac{F}{A} = \frac{\frac{1}{2}(1.917 \times 10^5 \text{ N})}{3.0 \times 10^{-4} \text{ m}^2} = \boxed{3.2 \times 10^8 \text{ N/m}^2} > 170 \times 10^6 \text{ N/m}^2$$

The stress is greater than the compressive strength, so the tibia will likely break.

2. As the brick falls, its potential energy is converted into kinetic energy. When the brick hits the floor, work is done on the brick to decelerate it to rest. The amount of work needed to decelerate the brick is equal to the initial potential energy (mgh) and is also equal to the product of the average stopping force (F) and the brick's compression distance ($\Delta\ell$). Use Eq. 9-4 to write the compression distance in terms of the force.

$$mgh = F\Delta\ell = F\left(\frac{1}{E} \frac{F}{A} \ell_0\right)$$

By replacing the strain (F/A) with the ultimate strength of brick, the resulting equation can be solved for the minimum height (h) necessary to break the brick when dropped.

$$h = \left(\frac{F}{A}\right)^2 \frac{\ell_0 A}{mgE} = \frac{(35 \times 10^6 \text{ N/m}^2)^2 (0.040 \text{ m})(0.150 \text{ m})(0.060 \text{ m})}{(1.2 \text{ kg})(9.80 \text{ m/s}^2)(14 \times 10^9 \text{ N/m}^2)} = \boxed{2.7 \text{ m}}$$

3. The ladder is in equilibrium, so both the net force and net torque must be zero. Because the ladder is on the verge of slipping, the static frictional force at the ground, F_{Cx} is at its maximum value. Thus, $F_{Cx} = \mu_s F_{Cy}$. Torques are taken about the point of contact of the ladder with the ground, and counterclockwise torques are taken as positive. The three conditions of equilibrium are as follows:

$$\sum F_x = F_{Cx} - F_W = 0 \rightarrow F_{Cx} = F_W$$

$$\sum F_y = F_{Cy} - Mg - mg = 0 \rightarrow$$

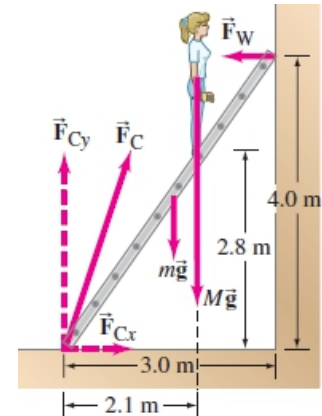
$$F_{Cy} = (M + m)g = (67.0 \text{ kg})(9.80 \text{ m/s}^2) = 656.6 \text{ N}$$

$$\sum \tau = F_W(4.0 \text{ m}) - mg\left(\frac{1}{2}\right)(3.0 \text{ m}) - Mg(2.1 \text{ m}) = 0$$

Solve the torque equation for F_W .

$$F_W = \left[\frac{\frac{1}{2}(12.0 \text{ kg})(3.0 \text{ m}) + (55.0 \text{ kg})(2.1 \text{ m})}{4.0 \text{ m}} \right] (9.80 \text{ m/s}^2) = 327.1 \text{ N}$$

The coefficient of friction then is then found from the components of F_C .



$$\mu_s = \frac{F_{Cx}}{F_{Cy}} = \frac{F_W}{F_{Cy}} = \frac{327.1 \text{ N}}{656.6 \text{ N}} = \boxed{0.50}$$